

EXPERIMENTAL MODELLING OF THE INFLUENCE OF VOCAL FOLDS COMPLIANCE ON HUMAN VOCAL TRACT ACOUSTIC PROPERTIES

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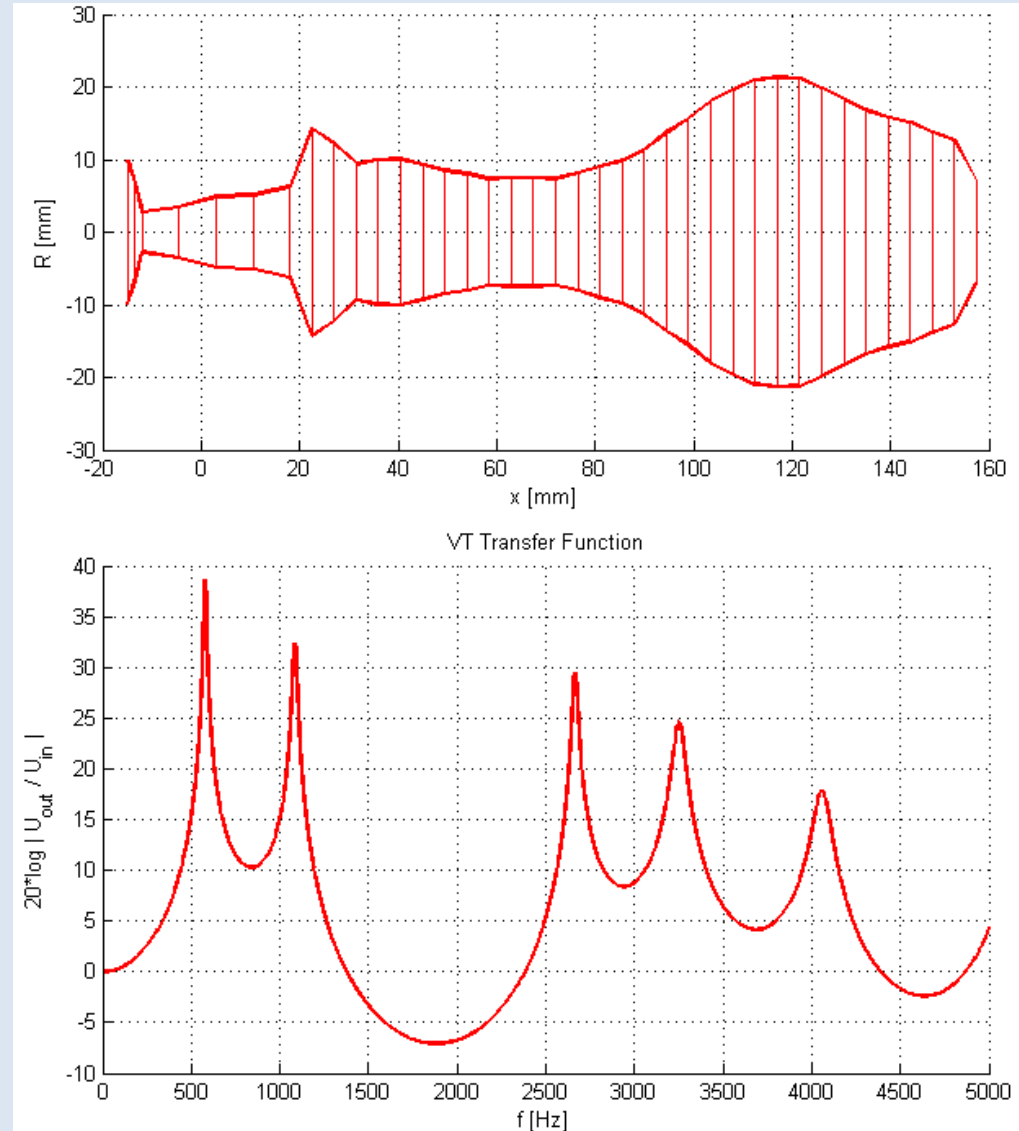
INTRODUCTION

Transfer function of VT

– describe acoustic resonance properties of this filter:
primary sound source --> voice sound radiated from the mouth.

– dependent on boundary conditions
(open x closed VF, closure done by soft tissue x hard wall).

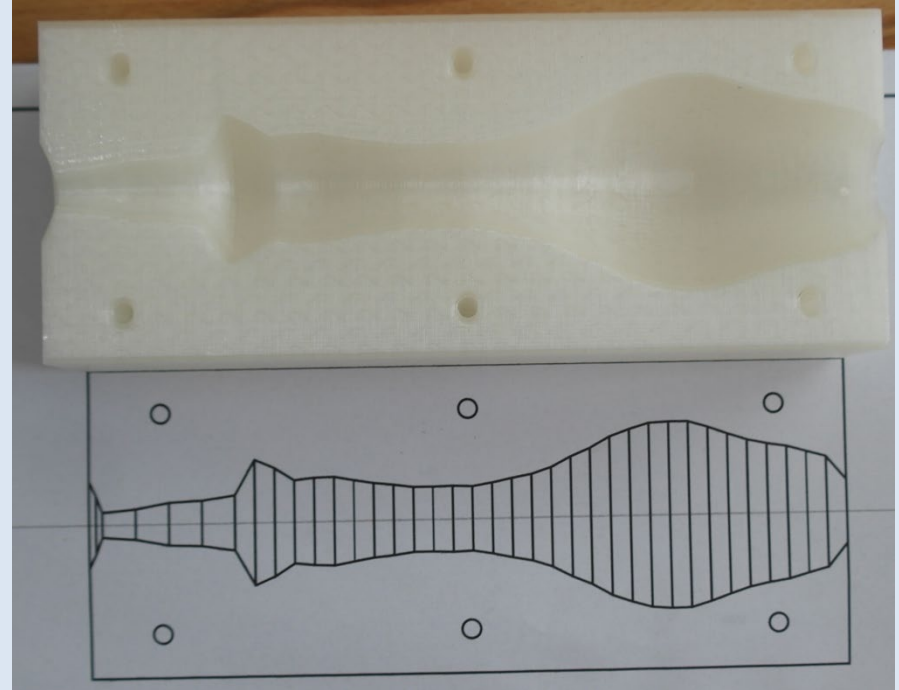
Modelling of these two problems properly resolved?



METHODS

3D VT model created from CT examination of a female subject during phonation [a:], see [1].

- simplified VT model with circular cross-sections was 3D printed using an acoustically hard material.

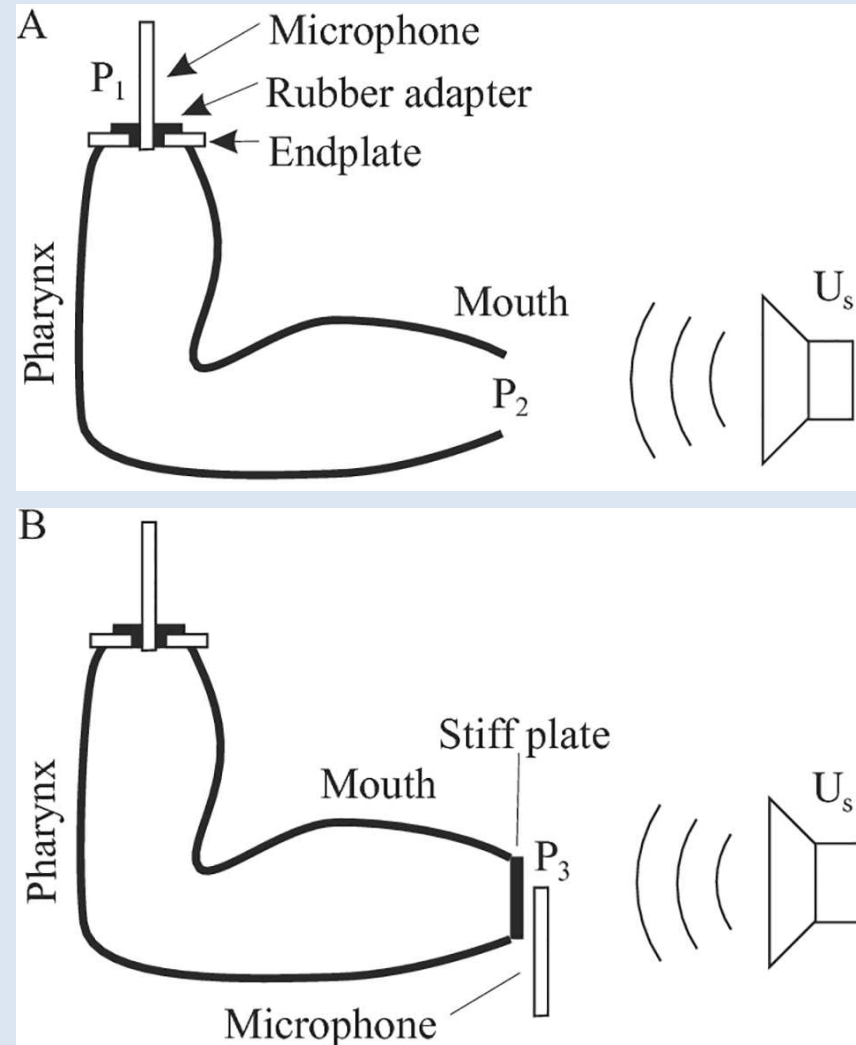


[1] Vampola T., Laukkanen A-M., Horáček J., Švec J.G.: Vocal tract changes caused by phonation into a tube: A case study using computer tomography and finite-element modeling. J. Acoust. Soc. Am. 129 (1), 2011, 310-315.

METHODS – MEASUREMENT SET UP

Volume velocity transfer function of the VT model was measured using the method described in [2]:

- excitation of the VT model with an external sound source in front of the lips,
- pressure P_1 is measured at the closed glottis while the mouth is open,
- pressure P_3 is measured right in front of the closed lips.
- $P_1/P_3 =$ volume velocity transfer function U_2/U_1

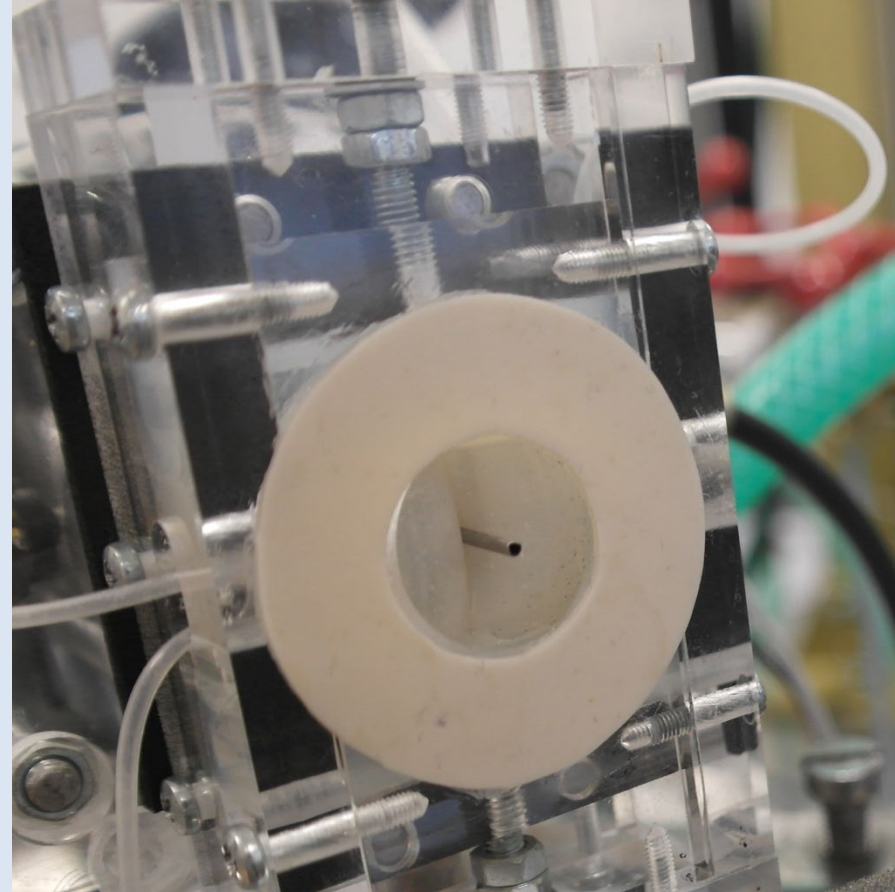


[2] Fleischer M., Mainka A., Kürbis S., Birkholz P.: How to precisely measure the volume velocity transfer function of physical vocal tract models by external excitation. PLoS ONE 13(3), 2018, e0193708.

METHODS – MEASUREMENT SET UP

We applied the experimental method [2] using the the three-layer model of vocal folds (silicone Ecoflex 00-10).

- Microphone probe inserted between the left and right part of VF model (connected to VT).
- Transfer function measured for several conditions of VF filled either with pressurized air or water.
- Loudspeaker 170 mm, 8 Ohm, 150 W, white noise signal.



[2] Fleischer M., Mainka A., Kürbis S., Birkholz P.: How to precisely measure the volume velocity transfer function of physical vocal tract models by external excitation. PLoS ONE 13(3): e0193708.

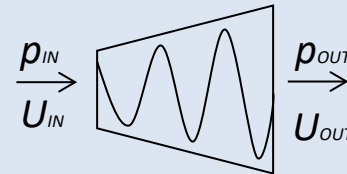
METHODS – MATHEMATICAL MODEL

Wave equation of an acoustic duct

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{1}{A} \cdot \frac{\partial A}{\partial x} \cdot \frac{\partial \phi}{\partial x} - \frac{1}{c_0^2} \cdot \left(\frac{\partial^2 \phi}{\partial t^2} + \frac{r_s}{\rho} \cdot \frac{\partial \phi}{\partial t} \right) = 0$$

Transfer matrix of a conical acoustic duct

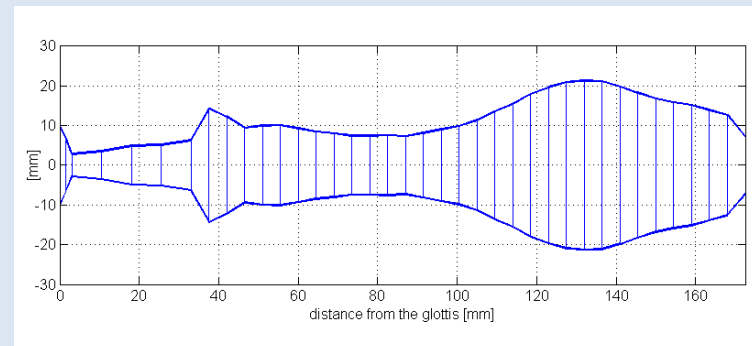
$$\begin{bmatrix} p_{OUT} \\ U_{OUT} \end{bmatrix} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \cdot \begin{bmatrix} p_{IN} \\ U_{IN} \end{bmatrix}$$



Transfer matrix of the vocal tract

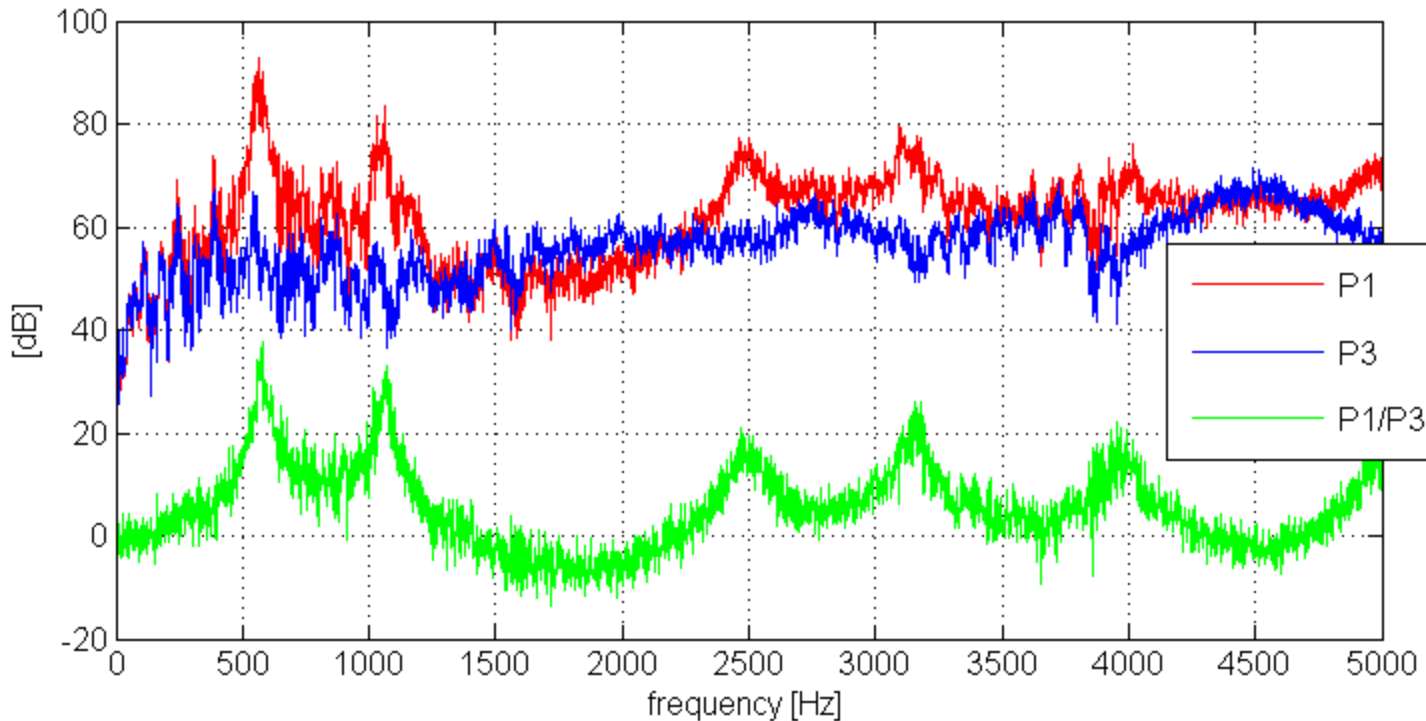
$$\begin{bmatrix} p_{LIP} \\ U_{LIP} \end{bmatrix} = \mathbf{T}_{VT} \cdot \begin{bmatrix} p_{GLOT} \\ U_{GLOT} \end{bmatrix}$$

$$\mathbf{T}_{VT} = \begin{bmatrix} a_{VT} & b_{VT} \\ c_{VT} & d_{VT} \end{bmatrix} = \mathbf{T}_{N_e+1, N_e} \cdot \mathbf{T}_{N_e, N_e-1} \cdot \dots \cdot \mathbf{T}_{3,2} \cdot \mathbf{T}_{2,1}$$



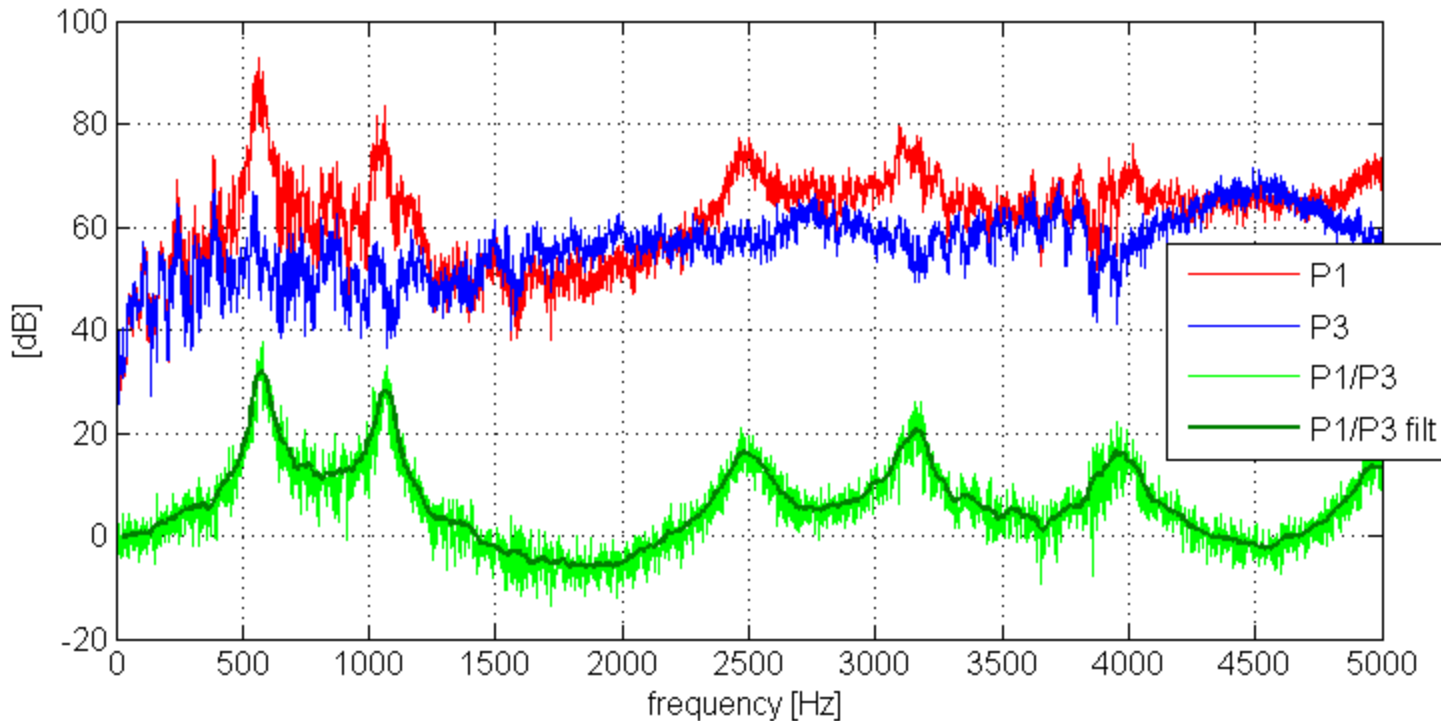
RESULTS

- Spectra of pressure signals and the transfer function
- hard wall closure at the glottis



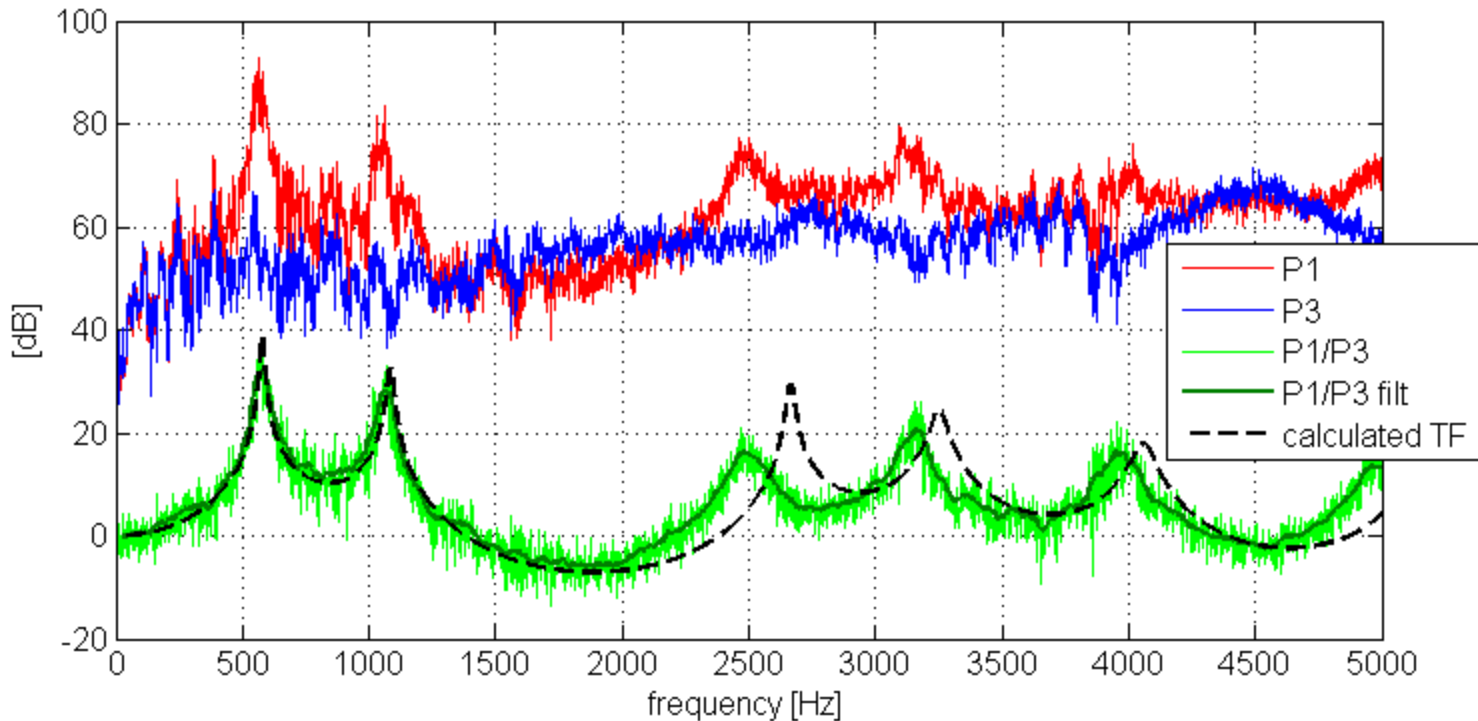
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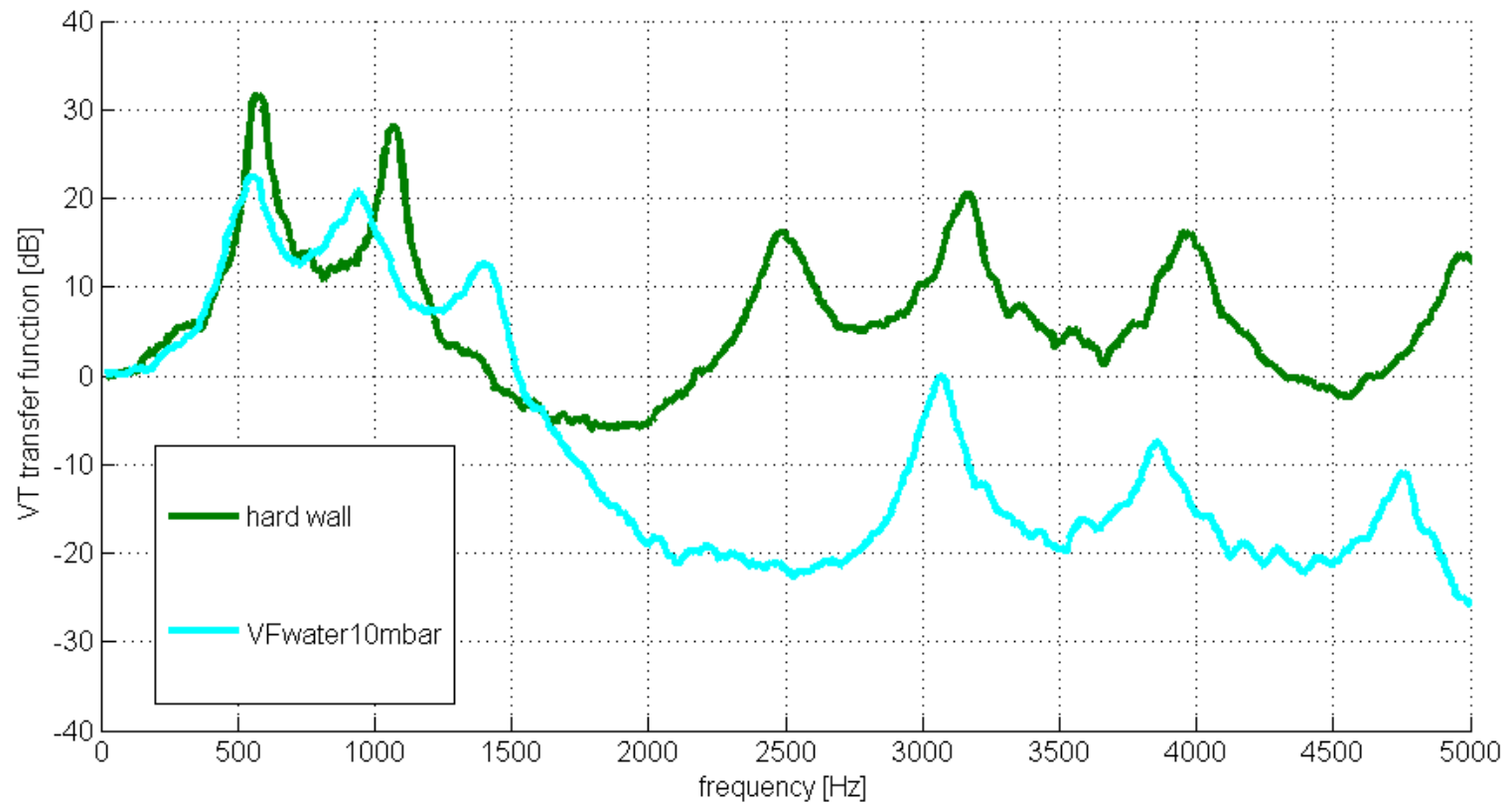
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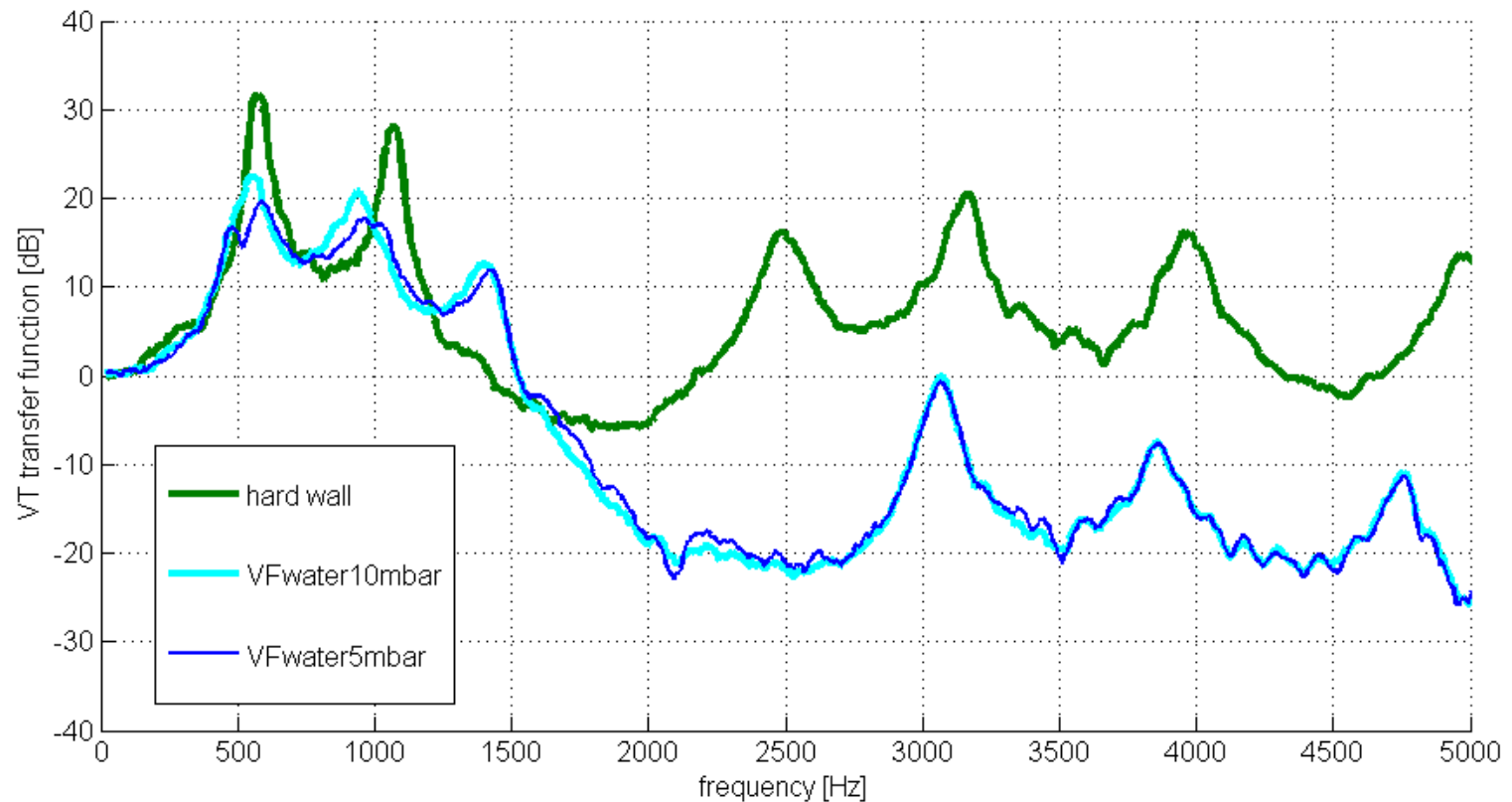
RESULTS

- The transfer function measured for different vocal folds stiffness



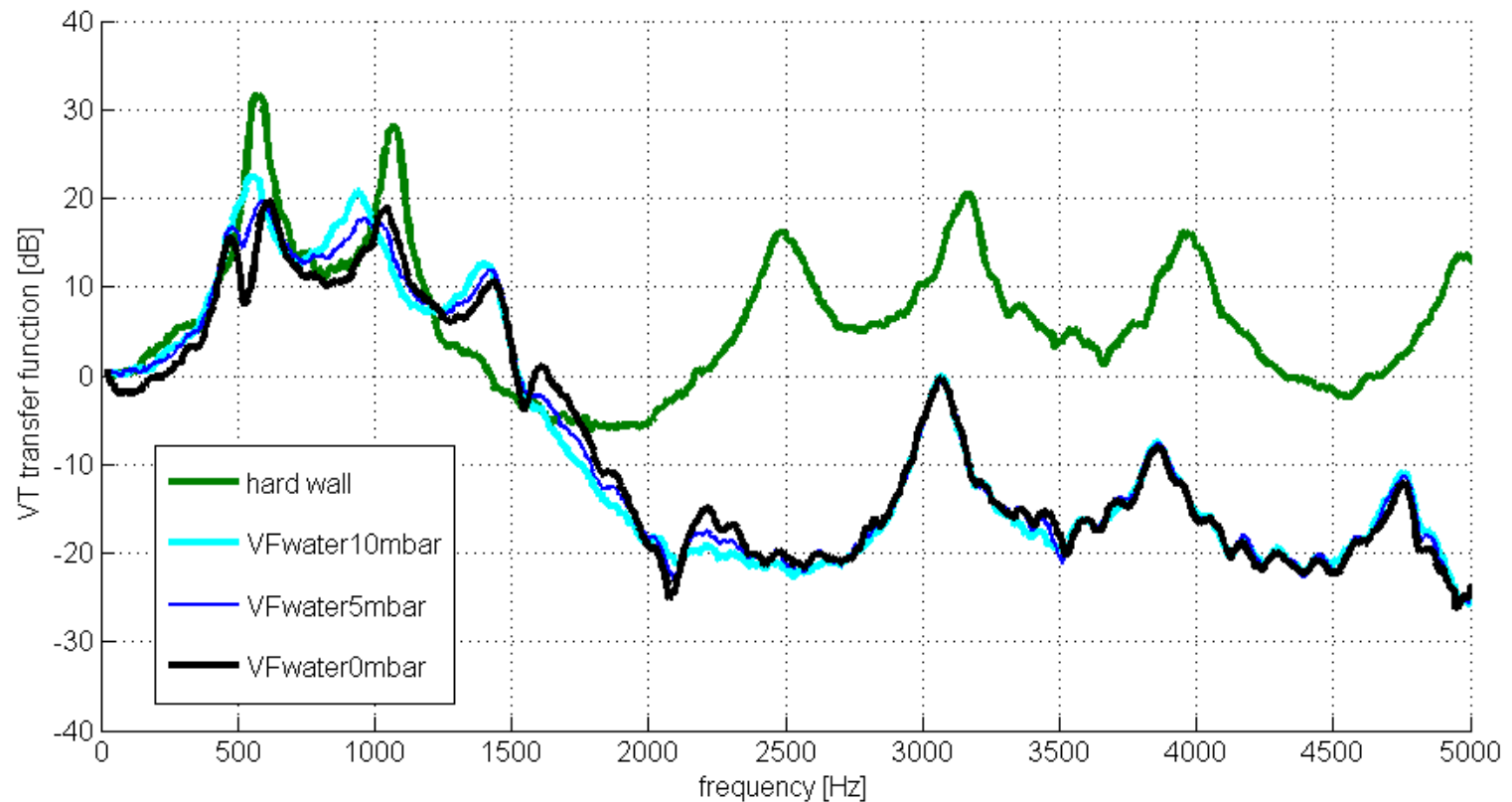
RESULTS

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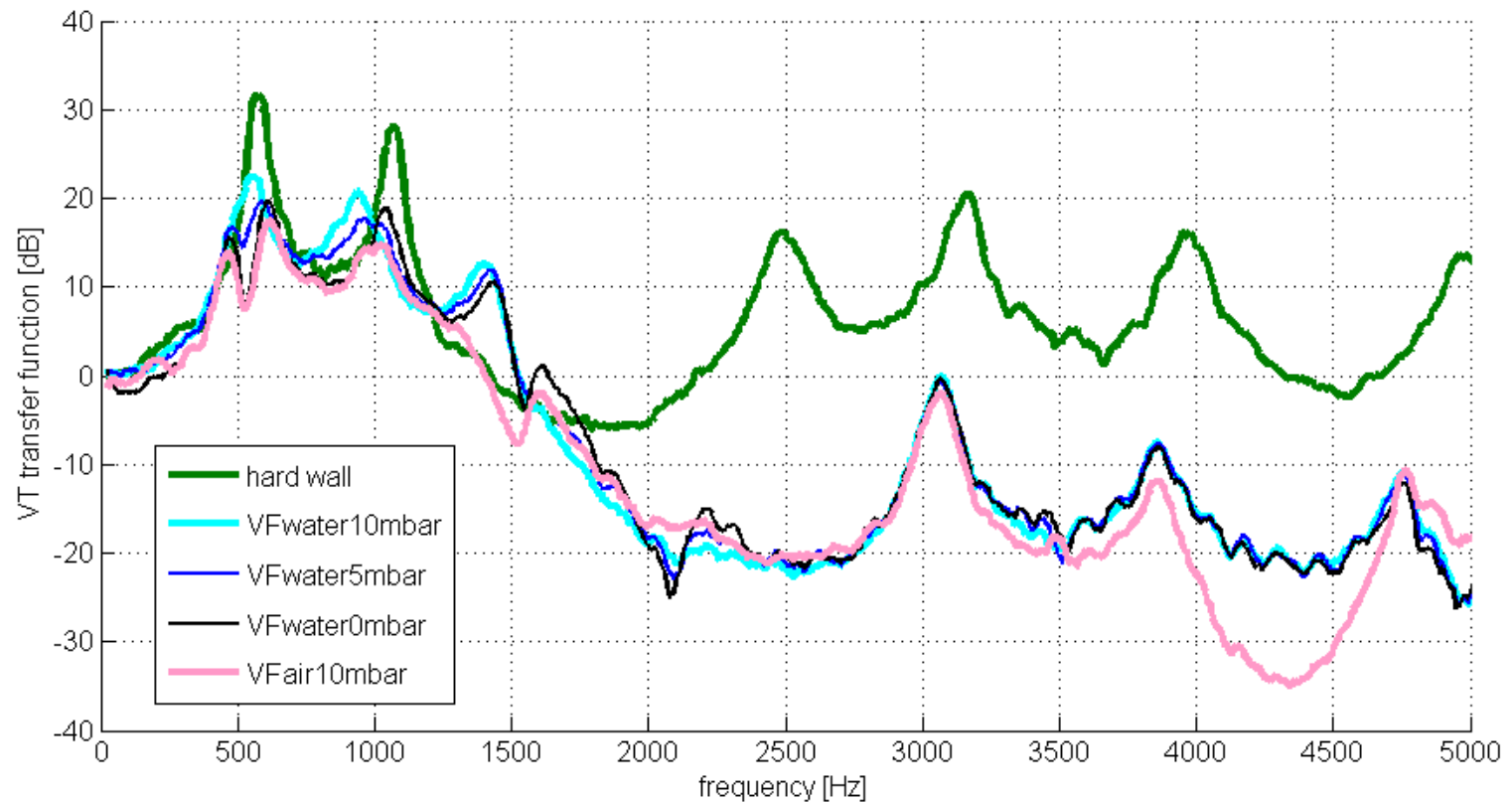
RESULTS

- The transfer function measured for different vocal folds stiffness



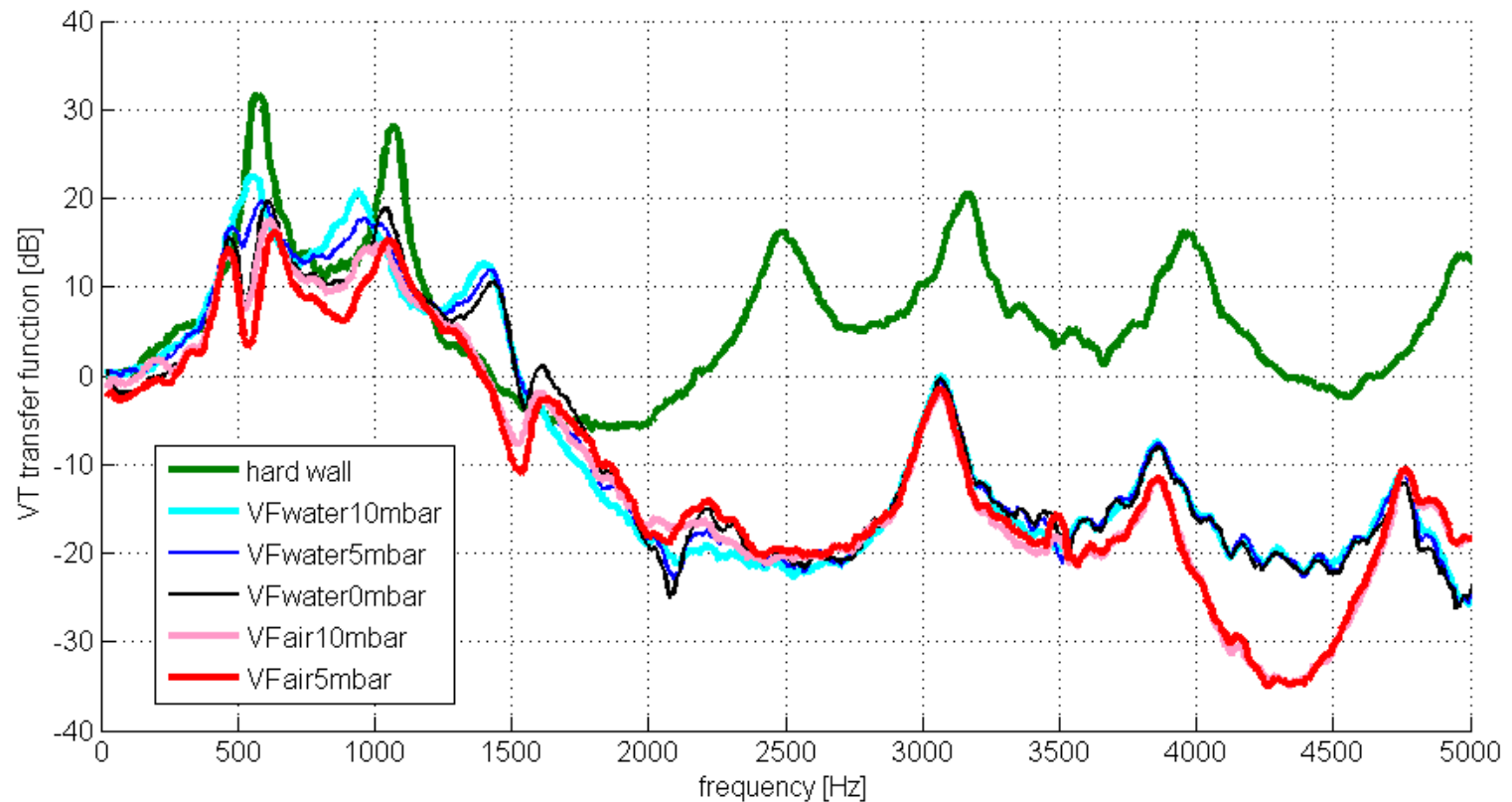
RESULTS

- The transfer function measured for different vocal folds stiffness



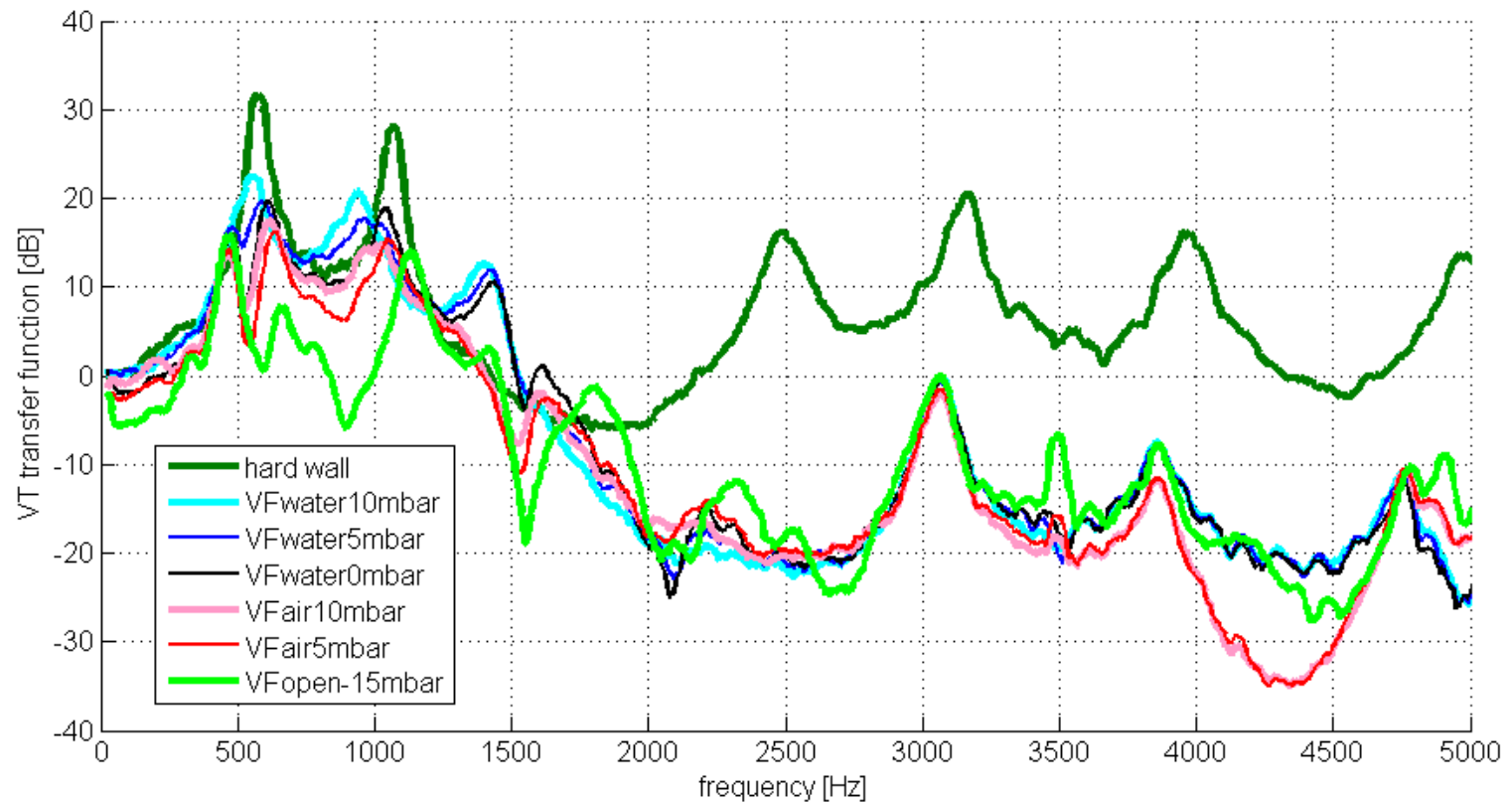
RESULTS

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RESULTS

- The transfer function measured for different vocal folds stiffness



DISCUSSION AND CONCLUSION

The calculated and measured transfer function of VT model with “hard-walled vocal folds” was compared. Calculated resonance frequencies differ from the measured ones by less than 7.2 %.

The stiffness and viscous properties of VFs can significantly change the VT frequency-modal and damping acoustic characteristics, especially in the frequency range above 2 kHz.

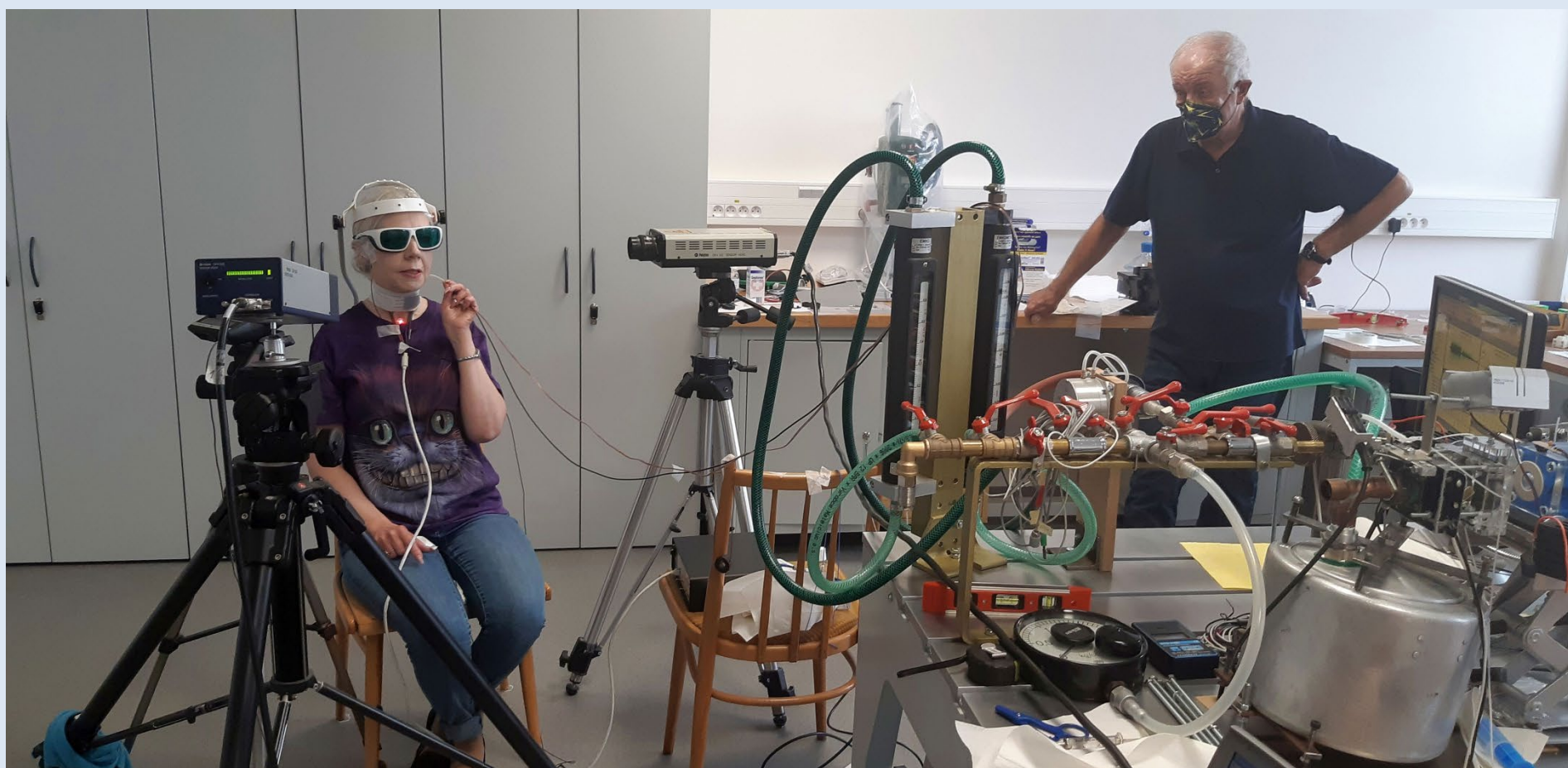
In future work, this phenomenon should be modelled mathematically.

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GREETINGS FROM CO-AUTHORS

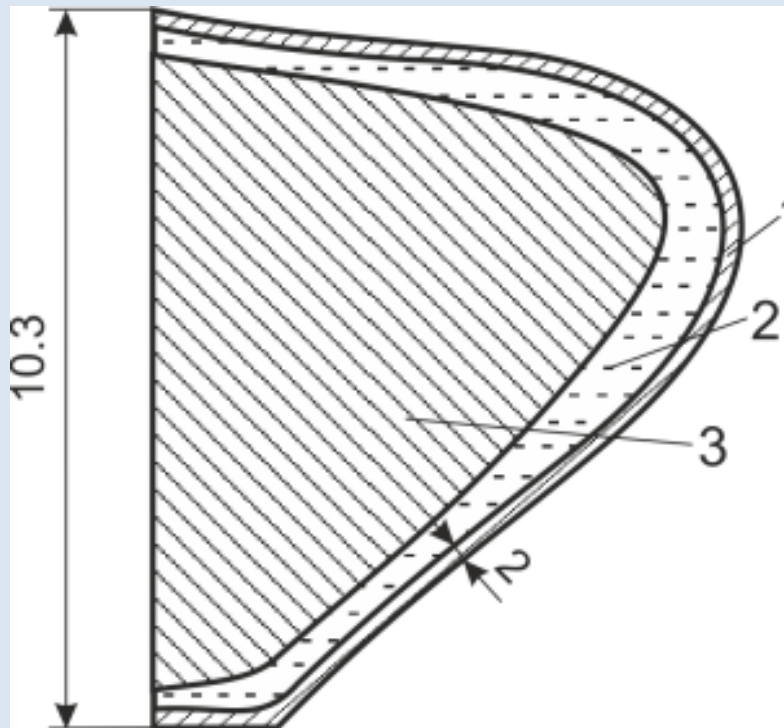
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Appendix - MEASUREMENT SET UP

1:1 scaled three layer vocal folds model



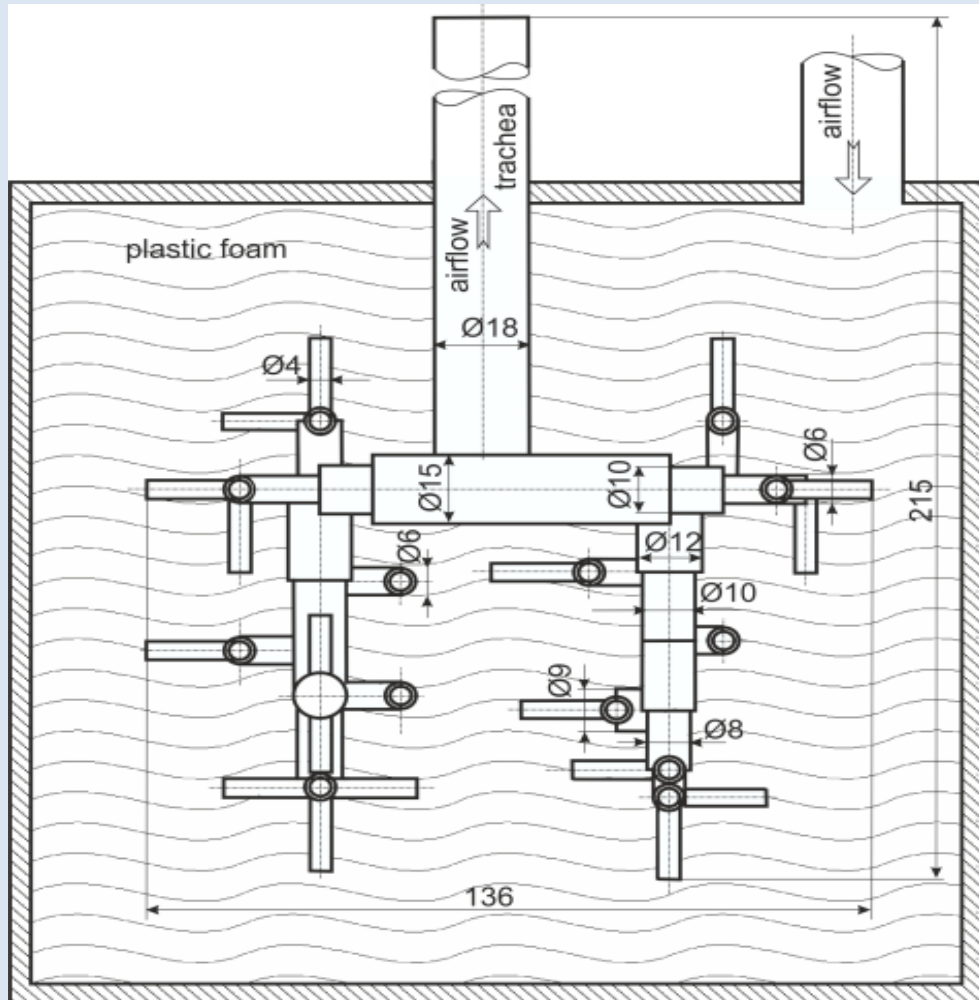
1 Thin silicon rubber cover

2 Liquid layer (lamina propria)

3 Silicon rubber wedge (vocal fold body)

Appendix - MEASUREMENT SET UP

- Schema of the lungs model



splitting of the airways
up to the
4th order branching

Appendix - 1D modelling

- ◆ Wave equation of an acoustic duct with a variable cross section $A(x)$

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{1}{A} \frac{\partial A}{\partial x} \frac{\partial \phi}{\partial x} - \frac{1}{c_0^2} \left(\frac{\partial^2 \phi}{\partial t^2} + \frac{r_s}{\rho} \frac{\partial \phi}{\partial t} \right) = 0$$

- velocity potential	ϕ	$[m^2 s^{-1}]$
- speed of sound	c_0	$[ms^{-1}]$
- specific acoustic resistance	$r_s = 2\pi\sqrt{f\mu\rho/A}$	$[kgm^{-3}s^{-1}]$
- fluid density	ρ	$[kg m^{-3}]$
- fluid dynamic viscosity	μ	$[kg m^{-1} s^{-1}]$
- frequency	f	$[Hz]$
<hr/>		
- acoustic pressure	$p = -\rho \frac{\partial \phi}{\partial t} - r_s \phi$	$[kgm^{-1}s^{-2} = Pa]$
- acoustic velocity	$v = \frac{\partial \phi}{\partial x}$	$[ms^{-1}]$
- volume velocity	$U = vA$	$[m^3 s^{-1}]$

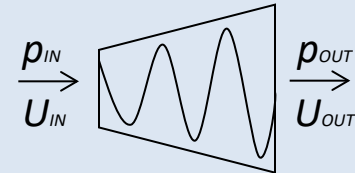
Appendix - 1D modelling

- 1D wave equation with variable cross-sectional area $A(x)$ and viscous damping.

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{1}{A} \cdot \frac{\partial A}{\partial x} \cdot \frac{\partial \phi}{\partial x} - \frac{1}{c_0^2} \cdot \left(\frac{\partial^2 \phi}{\partial t^2} + c_0 \cdot r_N \cdot \frac{\partial \phi}{\partial t} \right) = 0$$

- Analytical solution in frequency domain for a conical shape element.
- The form of transfer matrices for acoustic pressure p and volume velocity U .

$$\begin{bmatrix} p_{OUT} \\ U_{OUT} \end{bmatrix} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \cdot \begin{bmatrix} p_{IN} \\ U_{IN} \end{bmatrix} \quad \begin{array}{l} \text{(Radolf, 2010)} \\ \text{(Leino et al., 2011)} \end{array}$$



$$a = \frac{\xi_0}{\xi_0 + L} \cdot \left(\cosh(\gamma L) + \frac{1}{\gamma \xi_0} \cdot \sinh(\gamma L) \right)$$

$$b = -\frac{z_0(r_N + jk) \cdot \xi_0}{A_{IN} \cdot \gamma(\xi_0 + L)} \cdot \sinh(\gamma L)$$

$$c = A_{OUT} \cdot \frac{(1 - \gamma^2 \xi_0(\xi_0 + L)) \cdot \sinh(\gamma L) - \gamma L \cdot \cosh(\gamma L)}{\gamma(\xi_0 + L)^2 \cdot z_0(r_N + jk)}$$

$$d = \frac{A_{OUT}}{A_{IN}} \frac{\xi_0}{\xi_0 + L} \cdot \left(\cosh(\gamma L) - \frac{1}{\gamma(\xi_0 + L)} \cdot \sinh(\gamma L) \right)$$

$$\xi_0 = \frac{R_{IN}}{R_{OUT} - R_{IN}} \cdot L$$

$$r_N = \frac{1}{R} \cdot \sqrt{2k\mu/c_0\rho_0}$$

$$k = \omega/c_0$$

$$\gamma = \alpha + j\beta$$

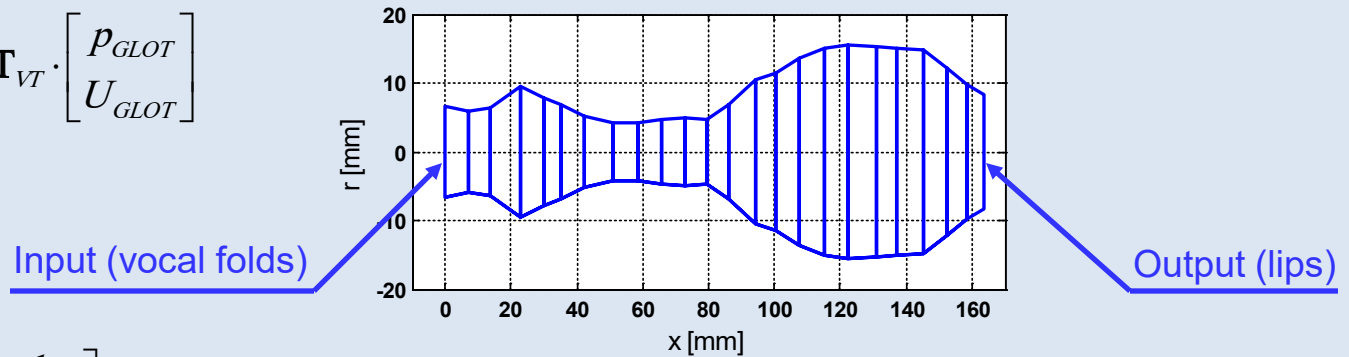
$$\alpha = \frac{r_N}{\sqrt{2 + 2 \cdot \sqrt{1 + (r_N/k)^2}}}$$

$$\beta = \frac{k}{2} \cdot \sqrt{2 + 2 \cdot \sqrt{1 + (r_N/k)^2}}$$

Appendix - 1D modelling

- Acoustic properties of the whole vocal tract

$$\begin{bmatrix} p_{LIP} \\ U_{LIP} \end{bmatrix} = \mathbf{T}_{VT} \cdot \begin{bmatrix} p_{GLOT} \\ U_{GLOT} \end{bmatrix}$$



$$\mathbf{T}_{VT} = \begin{bmatrix} a_{VT} & b_{VT} \\ c_{VT} & d_{VT} \end{bmatrix} = \mathbf{T}_{N_e+1, N_e} \cdot \mathbf{T}_{N_e, N_e-1} \cdot \dots \cdot \mathbf{T}_{3,2} \cdot \mathbf{T}_{2,1}$$

- Acoustic radiation impedance

$$Z_{Arad} = \frac{c_0 \rho_0}{\pi R^2} \cdot \left[1 - \frac{J_1(2kR)}{kR} + j \frac{H_1(2kR)}{kR} \right] = \frac{p_{LIP}}{U_{LIP}}$$

(Škvor 2001)

- Eigenfrequency calculation

$$U_{GLOT} = 0 \Rightarrow a_{VT} - Z_{Arad} \cdot c_{VT} = 0$$

- Acoustic pressure at the lips

$$p_{LIP} = \frac{a_{VT} \cdot d_{VT} - b_{VT} \cdot c_{VT}}{a_{VT} - Z_{Arad} \cdot c_{VT}} \cdot Z_{Arad} \cdot U_{GLOT}$$