



# YIELDING FUNCTIONS II

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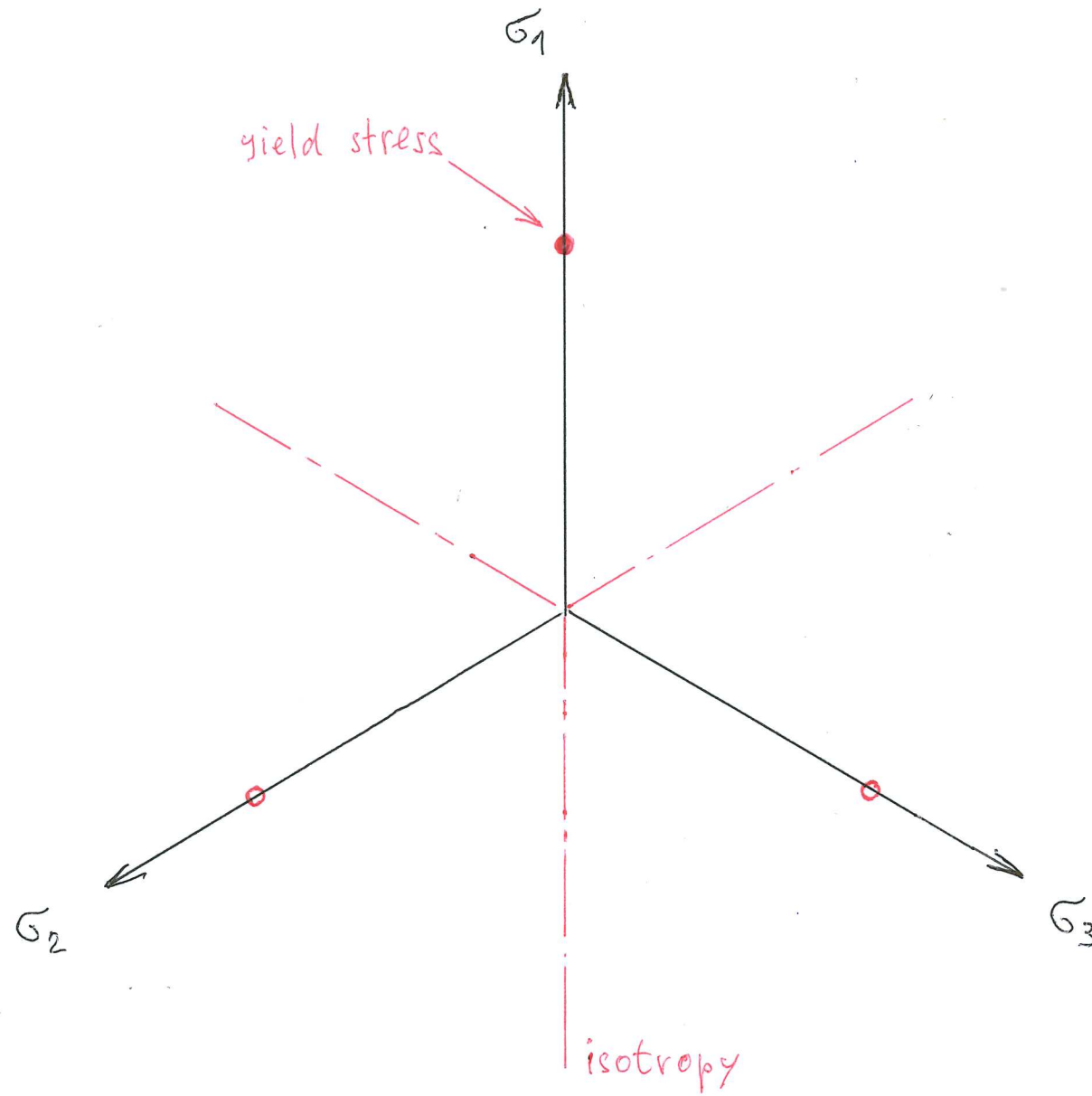
# Review

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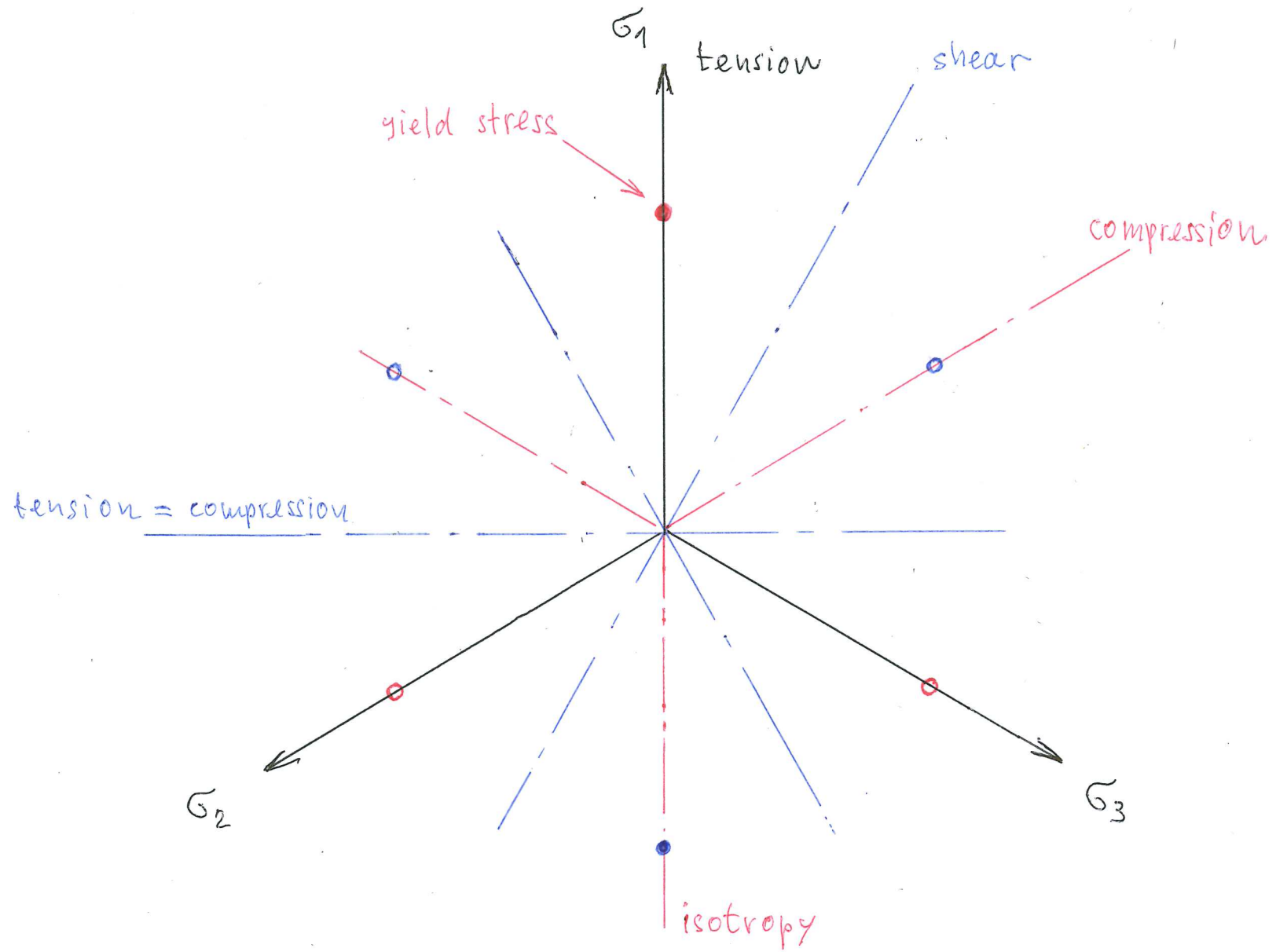
## Properties of yielding functions

- Convex (always)
- Deviatoric (often)
- Tension = compression (metals)
- Isotropy (often)

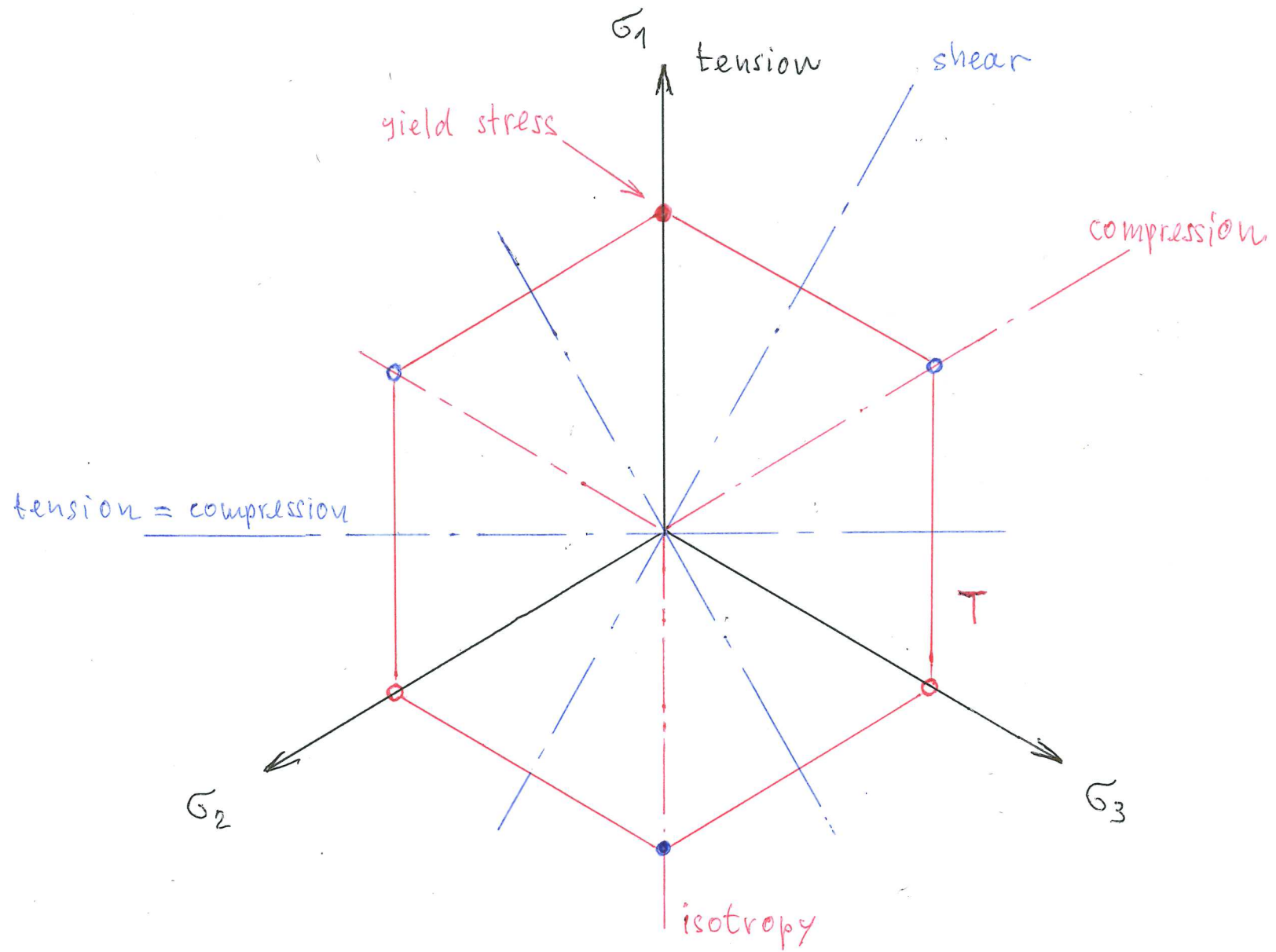
# Deviatoric plane



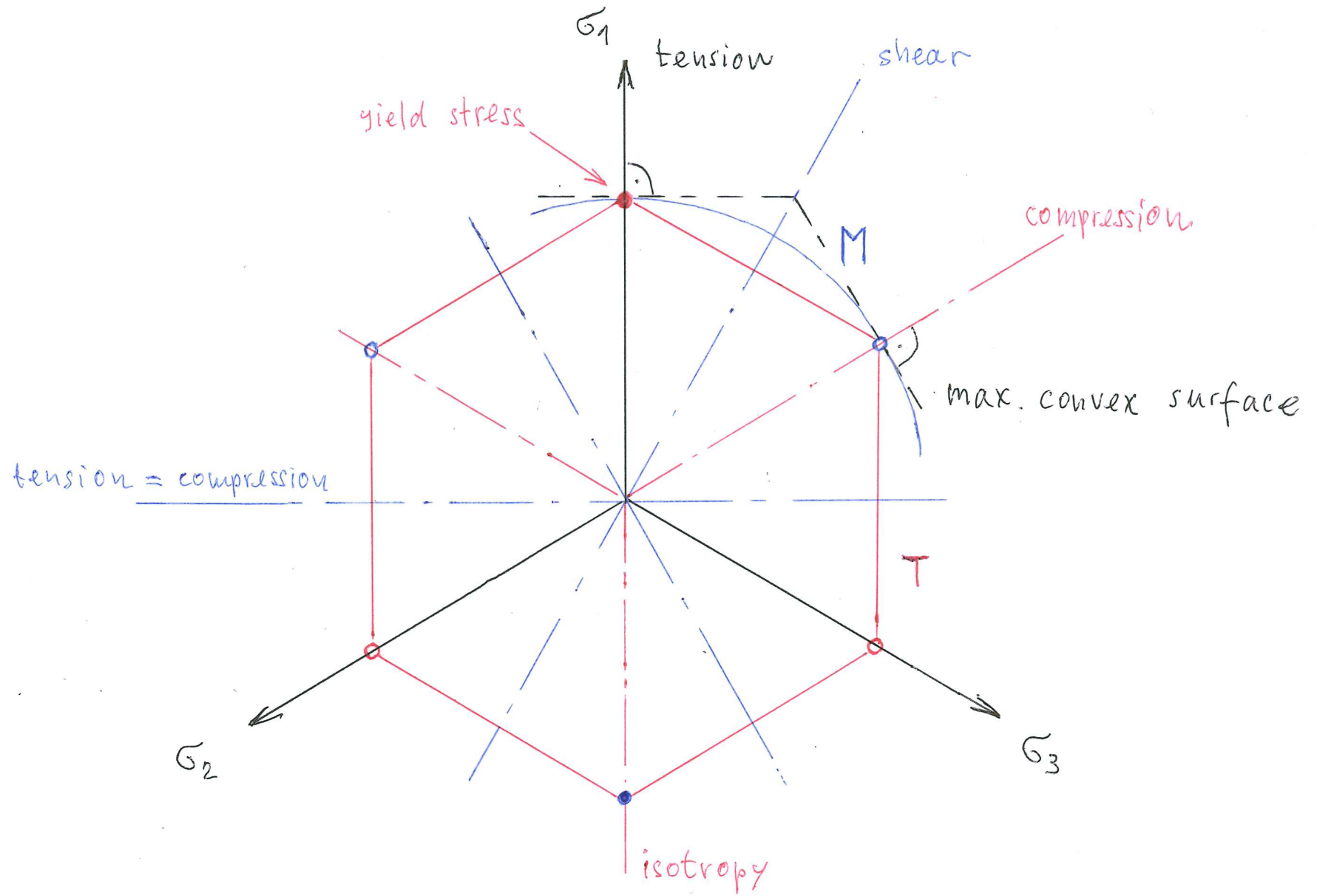
# Deviatoric plane



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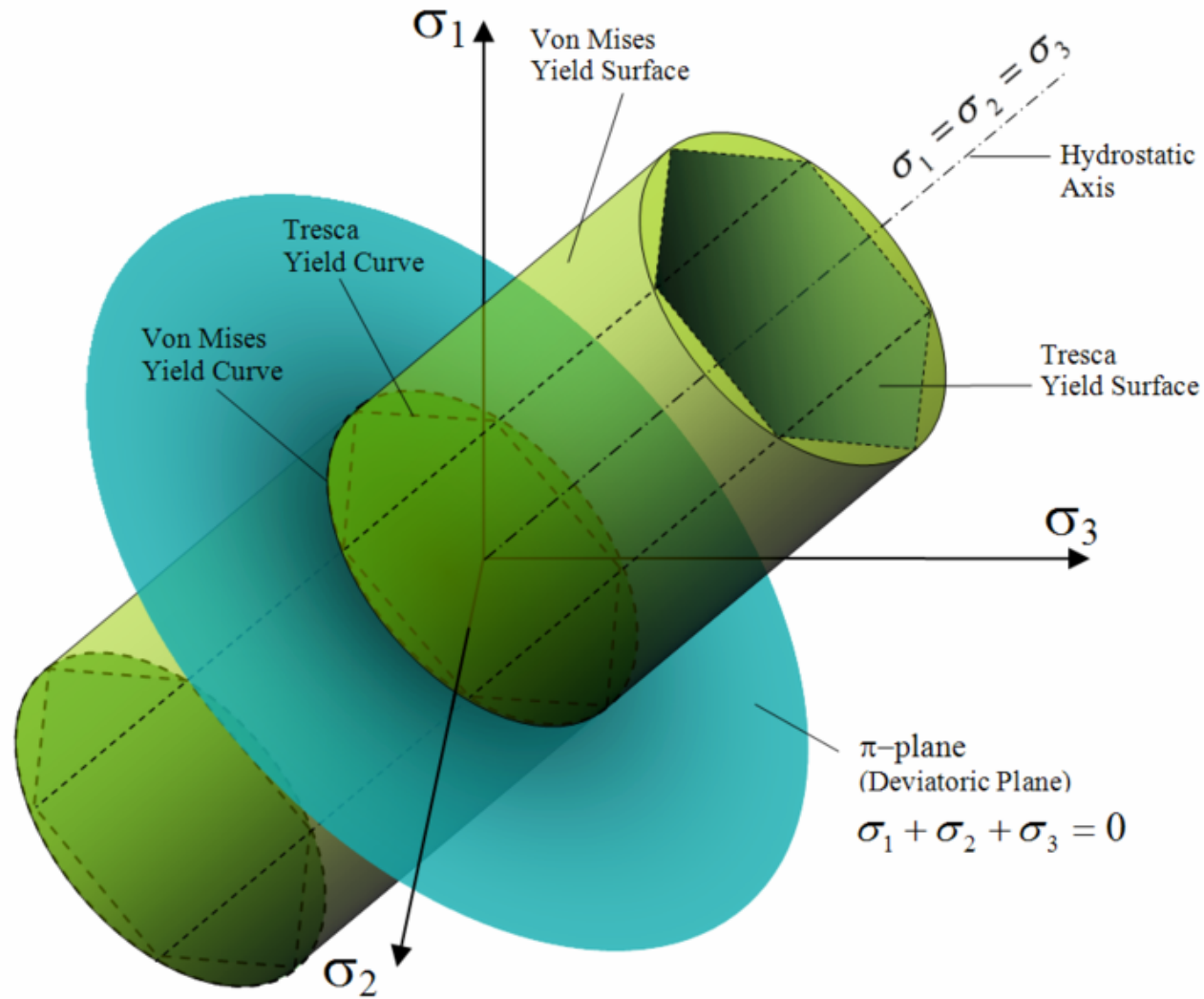


## Deviatoric plane



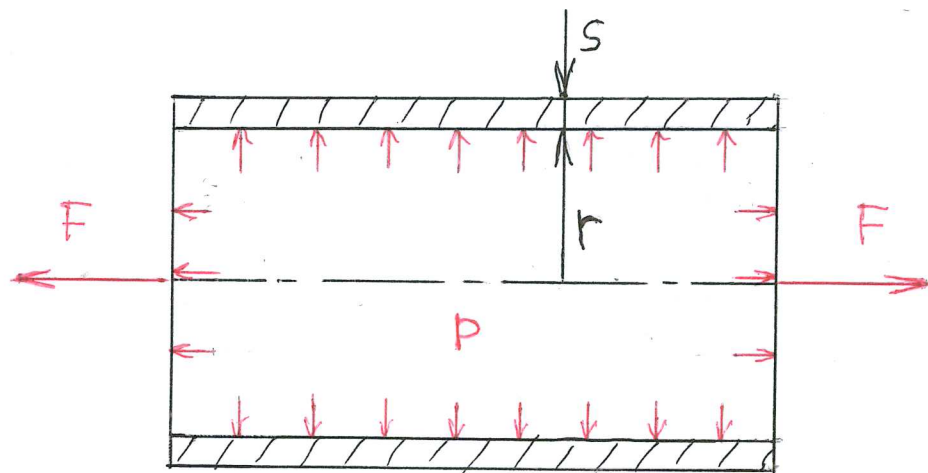
Remark: Tresca's condition is conservative.

# Tresca vs. von Mises





# Lode & Nādaï (1926)

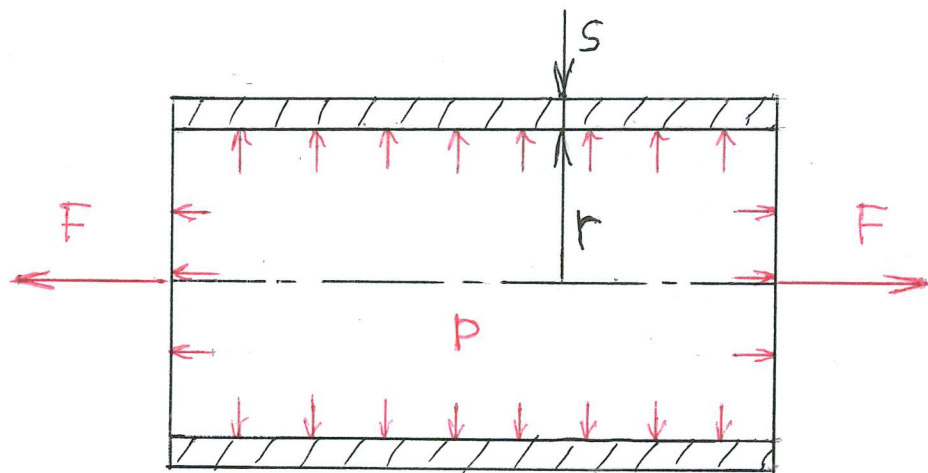


$$\sigma_t = \frac{pr}{s}$$

$$\sigma_o = \frac{pr}{2s} + \frac{F}{2\pi r s}$$

$$\sigma_r = 0 \quad (\text{at surface})$$

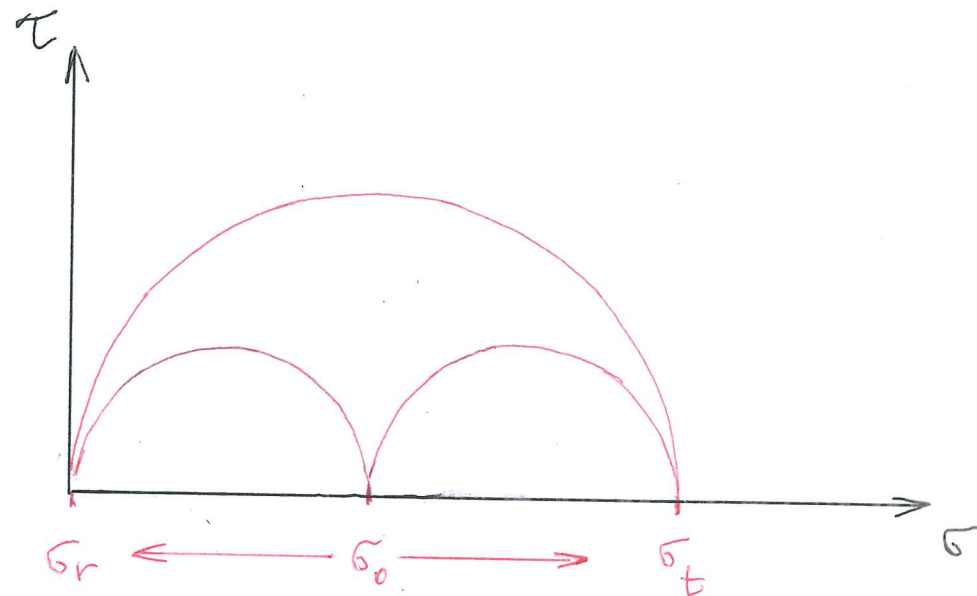
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$$-1 \text{ --- } \mu_\sigma \text{ --- } +1$$

$\mu_\sigma$  = Lode's parameter

(Experiments cover the whole sextant.)



# Lode's parameter

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Definition

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Exercise: Which of  $\sigma_r$ ,  $\sigma_o$ ,  $\sigma_t$  causes tension/shear/compression?

Remark: Biaxial tension generates uniaxial pressure!

Note: Any stress state is covered by varying  $\mu_\sigma$ .



# Prediction of yielding

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Tresca

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(independent of  $\mu_\sigma$ )



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...substituting to HMM

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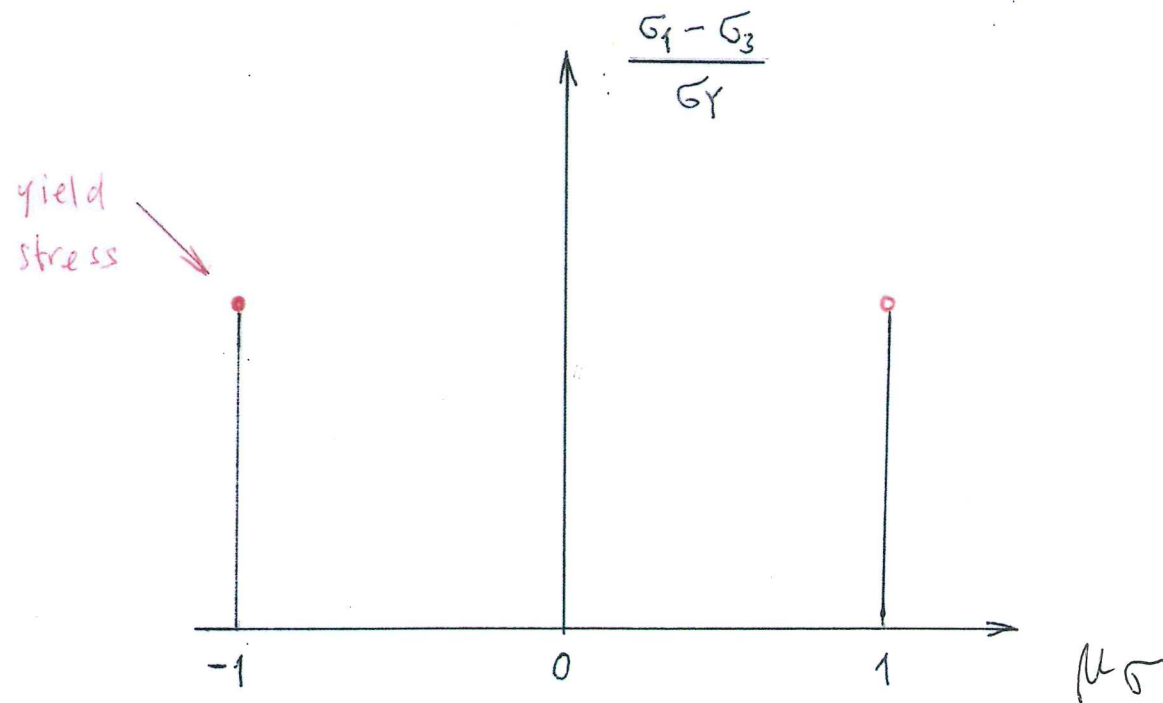
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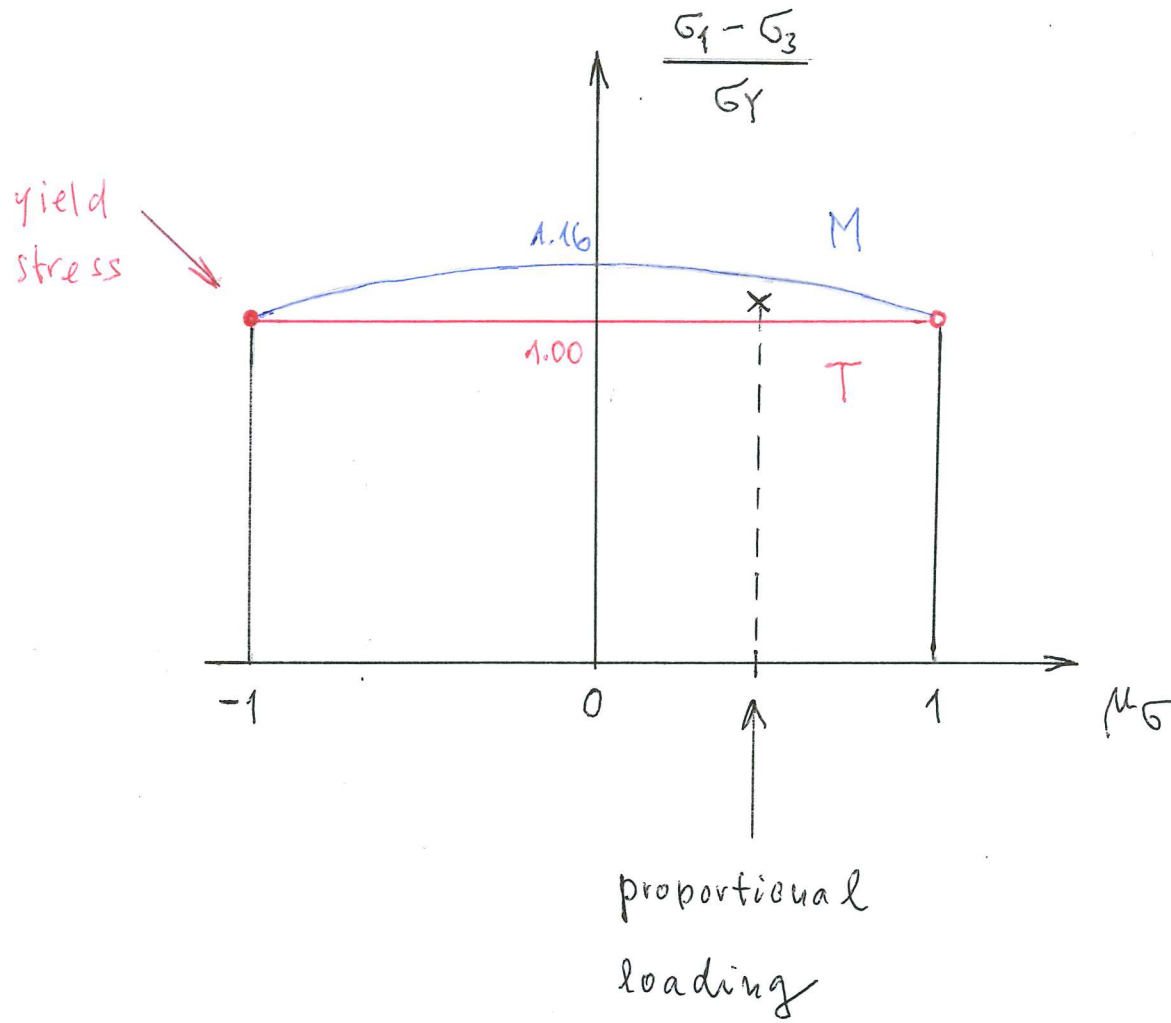
Property of Lode-Nádai setup: Load is controlled by  $p$ , yield limit by  $\mu_\sigma$ .

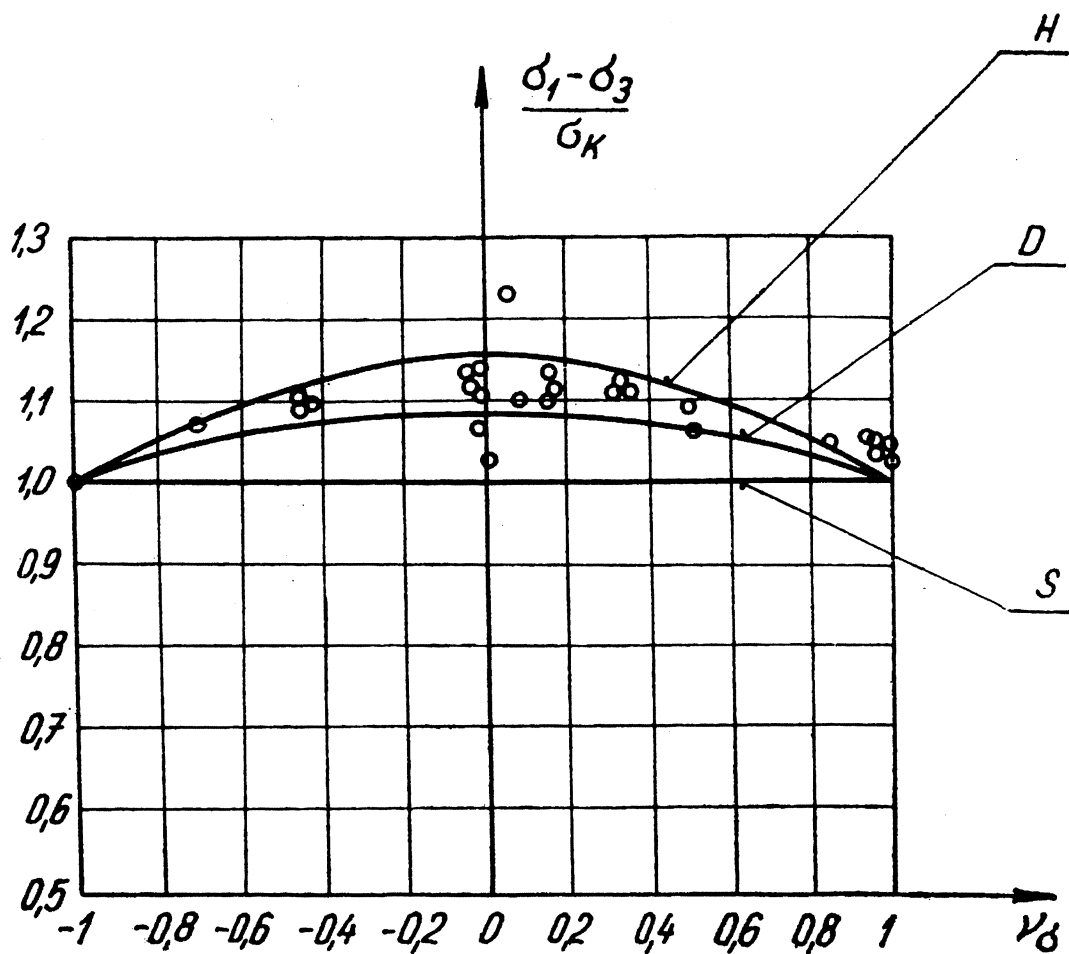


## Tresca versus Mises



## Tresca versus Mises





Průběhy hodnoty  $\frac{\sigma_1 - \sigma_3}{\sigma_K}$  podle různých podmínek plasticity a experimentální hodnoty Nádaie a Lodea



# Taylor & Quinney (1931)

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Tension-torsion of a thin wall tube

$$\sigma = \begin{bmatrix} \sigma & \tau \\ \tau & 0 \end{bmatrix}$$



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$$\det |\boldsymbol{\sigma} - \lambda \mathbf{I}| = \lambda^2 - \sigma\lambda - \tau^2 = 0$$



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$$\sigma_{1,3} = \frac{\sigma \pm \sqrt{\sigma^2 + 4\tau^2}}{2}$$



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Chart: ellipse equations



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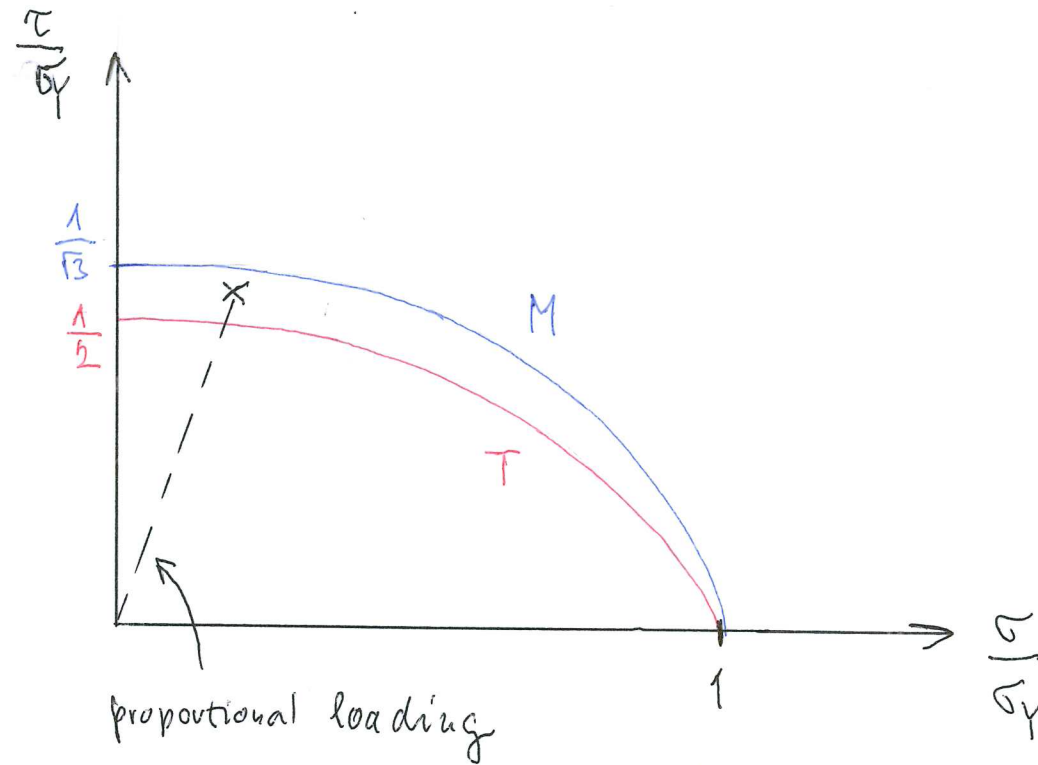
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Chart: ellipse equations

Remark: Only one twelfth of the deviatoric plane is covered.

## Tresca versus Mises



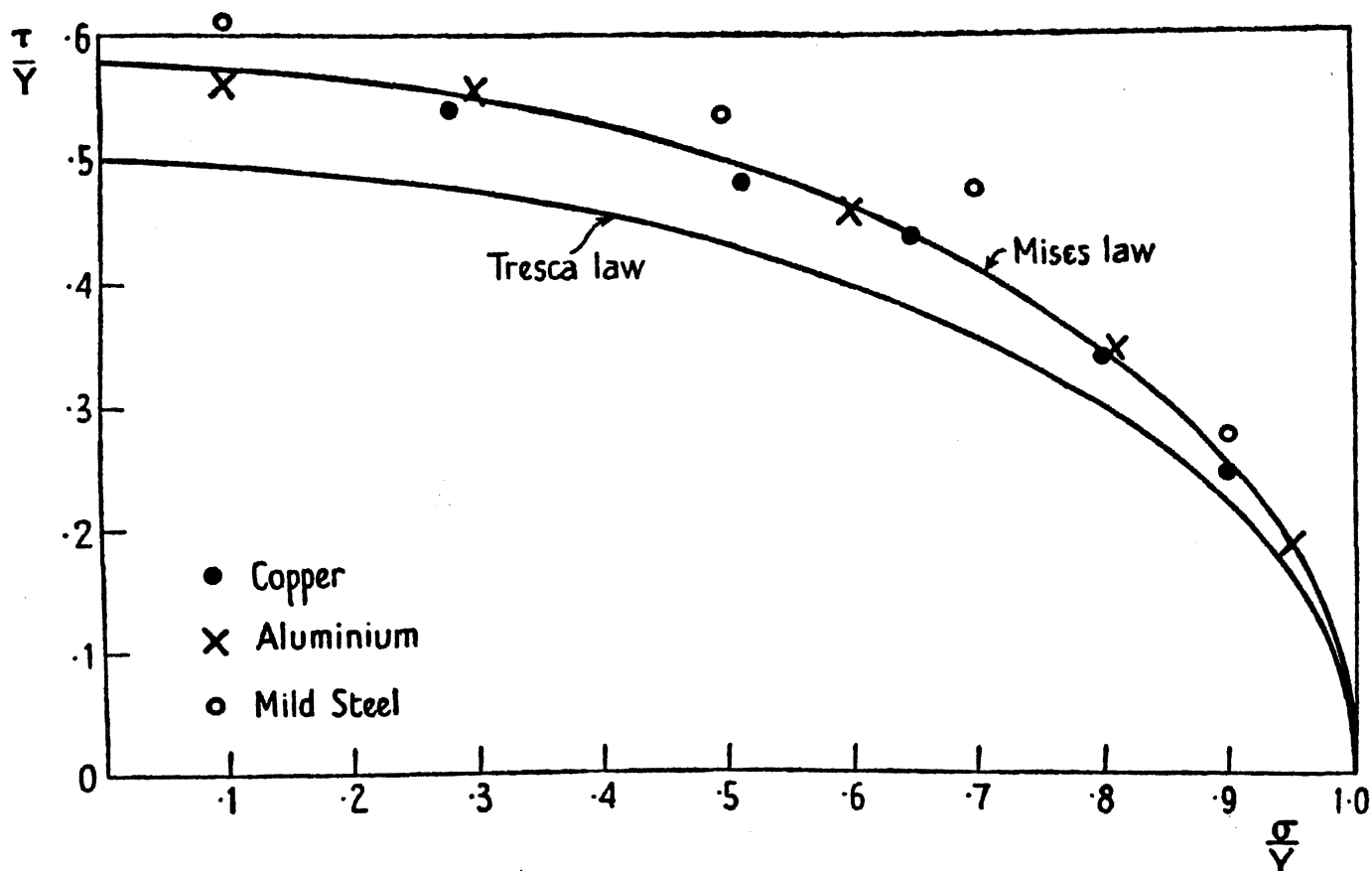


FIG. 4. Experimental results of Taylor and Quinney from combined torsion and tension tests, each metal being work-hardened to the same state for all tests. The Mises law is  $\sigma^2 + 3\tau^2 = Y^2$ , while the Tresca law is  $\sigma^2 + 4\tau^2 = Y^2$ , where  $\sigma$  = tensile stress,  $\tau$  = shear stress,  $Y$  = tensile yield stress.