



CONSERVATION LAWS II

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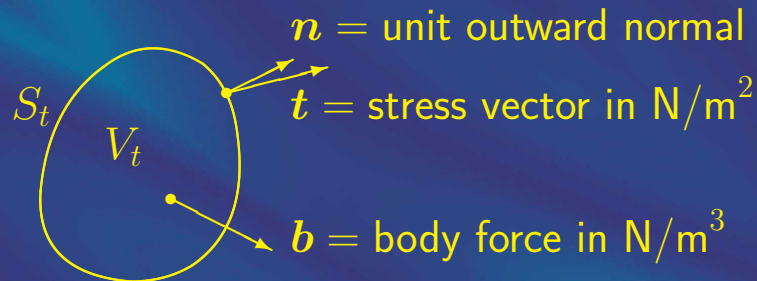
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- Conservation of momentum
 - Cauchy stress
 - 1st Piola-Kirchhoff stress
- Conservation of energy
 - Spatial description

Spatial description

In the current configuration ...

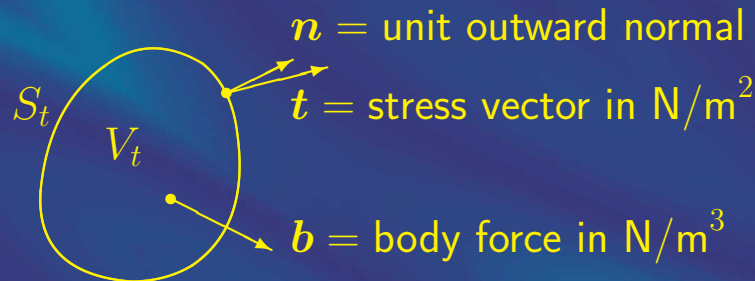
$$\frac{d\mathbf{p}}{dt} = \int_{V_t} \mathbf{b} dV_t + \int_{S_t} \mathbf{t} dS_t$$



Spatial description

In the current configuration ...

$$\frac{d\mathbf{p}}{dt} = \int_{V_t} \mathbf{b} dV_t + \int_{S_t} \mathbf{t} dS_t$$



By the Reynolds theorem ...

$$\frac{d}{dt} \int_{V_t} \rho \mathbf{v} dV_t = \int_{V_t} \rho \dot{\mathbf{v}} dV_t = \int_{V_t} \rho \mathbf{a} dV_t$$



Cauchy stress

The Cauchy theorem

$$\exists \boldsymbol{\sigma} \text{ in } \Omega_t : \mathbf{t} = \boldsymbol{\sigma} \mathbf{n}$$

- The stress tensor is defined in Ω_t independent of the choice of $S_t(V_t)$.



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$$\int_{S_t} t_i \, dS_t = \int_{S_t} \sigma_{ij} n_j \, dS_t = \int_{V_t} \frac{\partial \sigma_{ij}}{\partial x_j} \, dV_t$$



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Definition of the spatial divergence operation

$$\operatorname{div} \boldsymbol{\sigma} = \frac{\partial \sigma_{ij}}{\partial x_j} \mathbf{e}_i$$



Spatial equations of motion

Putting everything together ...

$$\int_{V_t} \rho \mathbf{a} \, dV_t = \int_{V_t} \mathbf{b} \, dV_t + \int_{V_t} \operatorname{div} \boldsymbol{\sigma} \, dV_t$$



Spatial equations of motion

Putting everything together ...

$$\int_{V_t} \rho \mathbf{a} \, dV_t = \int_{V_t} \mathbf{b} \, dV_t + \int_{V_t} \operatorname{div} \boldsymbol{\sigma} \, dV_t$$

It holds for every V_t , therefore

$$\operatorname{div} \boldsymbol{\sigma} + \mathbf{b} = \rho \mathbf{a} \quad \text{in } \Omega_t \equiv \Omega$$



Spatial equations of motion

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$$\int_{V_t} \rho \mathbf{a} \, dV_t = \int_{V_t} \mathbf{b} \, dV_t + \int_{V_t} \operatorname{div} \boldsymbol{\sigma} \, dV_t$$

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From the conservation of angular momentum it follows that $\boldsymbol{\sigma}$ is symmetric.



Example 5: Uniaxial stress



A = current crosssection

l = current length

$$\sigma_{11} = \frac{F}{A} \quad \text{true stress (Cauchy)}$$

$$P_{11} = \frac{F}{A_0} \quad \text{nominal stress (1st Piola-Kirchhoff)}$$

Example 5: Uniaxial stress



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$$P_{11} = \frac{F}{A_0} \quad \text{nominal stress (1st Piola-Kirchhoff)}$$

Idea: for $\Delta S_0 \subset S_0 \rightarrow \Delta S_t \subset S_t$ set the nominal stress vector \mathbf{T} such that

$$\int_{\Delta S_t} \mathbf{t} \, dS_t = \int_{\Delta S_0} \mathbf{T} \, dS_0$$



Transformation of inertial terms

Momentum

$$\frac{d\mathbf{p}}{dt} = \int_{V_t} \rho \mathbf{a} dV_t = \int_{V_0} J \rho \mathbf{a} dV_0$$



Transformation of inertial terms

Momentum

$$\frac{d\mathbf{p}}{dt} = \int_{V_t} \rho \mathbf{a} dV_t = \int_{V_0} \underbrace{J\rho}_{\rho_0} \mathbf{a} dV_0$$



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Body force

$$\int_{V_t} \mathbf{b} dV_t = \int_{V_0} \underbrace{J\mathbf{b}}_{\mathbf{B}} dV_0 = \int_{V_0} \mathbf{B} dV_0$$



Transformation of inertial terms

Momentum

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Body force in the reference configuration

$$\mathbf{B} = J\mathbf{b}$$



Material equations of motion

$$\int_{V_0} \rho_0 \mathbf{a} \, dV_0 = \int_{V_0} \mathbf{B} \, dV_0 + \int_{S_0} \mathbf{T} \, dS_0$$



Material equations of motion

$$\int_{V_0} \rho_0 \mathbf{a} \, dV_0 = \int_{V_0} \mathbf{B} \, dV_0 + \int_{S_0} \mathbf{T} \, dS_0$$

By the Cauchy theorem

$$\exists \mathbf{P} \text{ in } \Omega_0 : \mathbf{T} = \mathbf{P} \mathbf{N}$$

where \mathbf{N} is the unit outward normal respective to S_0 .



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where \mathbf{N} is the unit outward normal respective to S_0 .

Therefore

$$\text{Div } \mathbf{P} + \mathbf{B} = \rho_0 \mathbf{a} \quad \text{in } \Omega_0$$



Nanson's formula

Given a general field \mathbf{T} = scalar, vector, tensor, ...

$$\mathbf{T} = \phi(\mathbf{X}, t) = \varphi(\mathbf{x}, t)$$

It holds

$$\int_{\Delta S_t} \mathbf{T} \mathbf{n} \, dS_t = \int_{\Delta S_0} J \mathbf{T} \mathbf{F}^{-T} \mathbf{N} \, dS_0$$



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Tentative proof for a closed surface

$$\int_{S_t} \varphi n_j \, dS_t =$$



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$$\int_{S_t} \varphi n_j \, dS_t = \int_{V_t} \frac{\partial \varphi}{\partial x_j} \, dV_t$$



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$$\int_{S_t} \varphi n_j \, dS_t = \int_{V_t} \frac{\partial \varphi}{\partial x_j} \, dV_t = \int_{V_0} J \frac{\partial \phi}{\partial X_k} \frac{\partial X_k}{\partial x_j} \, dV_0$$

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Gabrio Piola (1794-1850): Piola identity (underlined), Piola transformation



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1st Piola-Kirchhoff stress tensor

Component notation

$$\int_{\Delta S_t} t_i \, dS_t =$$



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$$\int_{\Delta S_t} t_i \, dS_t = \int_{\Delta S_t} \sigma_{ij} n_j \, dS_t$$



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Definition

$$P_{ik} = J \sigma_{ij} F_{kj}^{-1} \Rightarrow \boxed{\mathbf{P} = J \boldsymbol{\sigma} \mathbf{F}^{-T}} \quad \text{nonsymmetric}$$



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Definitions

$$P_{ik} = J \sigma_{ij} F_{kj}^{-1} \quad \Rightarrow \quad \boxed{\mathbf{P} = J \boldsymbol{\sigma} \mathbf{F}^{-T}} \quad \text{nonsymmetric}$$

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$$T_i = P_{ik} N_k \quad \Rightarrow \quad \mathbf{T} = \mathbf{P} \mathbf{N}$$

- Note that the Cauchy relation was derived by transformation.



4. Conservation of energy

$$\dot{Q} + \dot{W} = \frac{d}{dt}(U + K)$$



Kinetic energy

Owing to Reynold's theorem ...

$$\frac{d}{dt}K = \frac{d}{dt} \frac{1}{2} \int_{V_t} \rho \|\mathbf{v}\|^2 dV_t$$



Kinetic energy

Owing to Reynold's theorem ...

$$\frac{d}{dt}K = \frac{d}{dt} \frac{1}{2} \int_{V_t} \rho \|\mathbf{v}\|^2 dV_t = \frac{1}{2} \int_{V_t} \rho (\mathbf{v} \cdot \mathbf{v})^\cdot dV_t$$



Kinetic energy

Owing to Reynold's theorem ...

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Kinetic energy

Owing to Reynold's theorem ...

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Kinetic energy

Owing to Reynold's theorem ...

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Kinetic energy

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Thus, we have

$$\frac{d}{dt}K = \int_{V_t} \rho \mathbf{a} \mathbf{v} dV_t$$



Mechanical power (1/2)

Definition

$$\dot{W} = \int_{V_t} \mathbf{b} \cdot \mathbf{v} \, dV_t + \int_{S_t} \mathbf{t} \cdot \mathbf{v} \, dS_t$$



Mechanical power (1/2)

Definition

$$\dot{W} = \int_{V_t} \mathbf{b} \cdot \mathbf{v} \, dV_t + \int_{S_t} \mathbf{t} \cdot \mathbf{v} \, dS_t$$

Surface integral

$$\int_{S_t} \mathbf{t} \cdot \mathbf{v} \, dS_t = \int_{S_t} t_i v_i \, dS_t$$



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Inserting ...

$$\dot{W} = \int_{V_t} (\mathbf{b} + \operatorname{div} \sigma) \cdot \mathbf{v} \, dV_t + \int_{V_t} \sigma_{ij} L_{ij} \, dV_t$$

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$$\begin{aligned} \int_{S_t} \mathbf{t} \cdot \mathbf{v} \, dS_t &= \int_{S_t} t_i v_i \, dS_t = \int_{S_t} \sigma_{ij} n_j v_i \, dS_t = \int_{V_t} \frac{\partial}{\partial x_j} (\sigma_{ij} v_i) \, dV_t \\ &= \int_{V_t} \frac{\partial \sigma_{ij}}{\partial x_j} v_i \, dV_t + \int_{V_t} \sigma_{ij} \frac{\partial v_i}{\partial x_j} \, dV_t = \int_{V_t} \mathbf{v} \cdot \operatorname{div} \sigma \, dV_t + \int_{V_t} \sigma_{ij} L_{ij} \, dV_t \end{aligned}$$

Inserting ...

$$\dot{W} = \int_{V_t} \underbrace{(\mathbf{b} + \operatorname{div} \sigma)}_{\rho \mathbf{a}} \cdot \mathbf{v} \, dV_t + \int_{V_t} \sigma_{ij} L_{ij} \, dV_t = \frac{d}{dt} K + \int_{V_t} \sigma_{ij} L_{ij} \, dV_t$$



Mechanical power (2/2)

Additive decomposition

$$\sigma_{ij}L_{ij} = \sigma_{ij}(D_{ij} + W_{ij}) = \sigma_{ij}D_{ij}$$



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At $t = 0$: $\boldsymbol{\sigma}:\mathbf{D} = \boldsymbol{\sigma}:\dot{\boldsymbol{\epsilon}} = \sigma_{ij}\dot{\epsilon}_{ij}$ (small strain approximation)



Spatial equation of heat conduction

Thus, even for fast loadings ...

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Heat conduction with a thermomechanical coupling term

$$\kappa - \operatorname{div} \mathbf{h} + \boldsymbol{\sigma} : \mathbf{D} = \rho \dot{u} \quad \text{in } \Omega_t \equiv \Omega$$